# AUTOBEAM AN EXPERT SYSTEM TO ANALYSE AND DESIGN R.C.C. BEAMS 

Dissertation submitted to the Jawaharlal Nehru University in partial fulfilment of the requirements for the award of the Degree of MASTER OF TECHNOLOGY

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## AUTO BEAM

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The quest to automate the civil engineering design process has lead the designer to opt for computers. Civil engineering design problems characterised by their inherent impression, paucity incompleteness of data, and heavy relience on expert views, defys the algorithmic approach, since the experience and intutive judgement form an important role in the design process. So the need of a software which posseses the knowledge of an expert is inevitable, which guides the designer towards the solution.

The Expert systems are the computer programs which were built into the knowledge and are capable to operate at the expert level. The Expert System naturally contains large and varied knowledge about, a specific area, from which the inference mechanism infers the context based inferences.

PROLOG has a inbuilt backward chaining inference mechanism, capable of representing the fuzzy and imprecise knowledge and capble of exploring the parallel processing architecures; promotes it to develop Expert Systems.

The problem of designing the reinforced concrete beams demands risorous numerical computation to analyse the beato and a knowledge base containing the knowledge and the rules, formulae and specifications of design, and a inference mechanism which infers context based eonclusions.

It is not easy to implement both the symbolie and numerical computations using a single prosraming language. PROLOG is used to implement the symbolie computations and PASCAL used to implement the numerical eomputations. The linking and data transfer is through l \% files. The nemerical computations routine implemented in PASCAL is executed in PROLOG environment.

This program posseses all the kroowledge needed to analyse and design any of the cantilever, simply supported, over hangining and fixed beams with any number of point or uniformly distributed loads.

Analysis part of the program implemented in PASCAL analyses all the loads and displays the output graphically and transfers the data into a file. This data file is read in PROLOG environment and the data is stored in the knowledge base as context based facts. The design, implemented in PROLOG is based on the IS-456 codal specifications. The section of the beam and the reinforcement details which are satisfactory to the user and not contradicting the code is arrived at. All the drawing are displayed graphically.

## CONTENTS

1. THE NEED OF EXPERT SYSTEMS.
2. INTRODUCTION TO EXPERT SYSTEMS.
3. SUTABILITY OF PROLOG TO IMPLEMENT EXPERT SYSTEMS.
4. RULES IN THE KNOWLEDGE BASE.
5. THEORY OF DESIGN.
6. IMPLEMENTATION.
6.1 : IMPLEMENTATION OF ANALYSIS.
7. 2 : LINKING PASCAL AND PROLOG.
6.3 : IMPLEMENTATION OF DESIGN.
6.4 : GRAPHICAL DISPLAY.
8. PROGRAM LISTINGS.

## CHAPTER.1. THE NEED OF EXPERT SYSTEM

The present civil engineer is confronted with new challenges in the design process due to the increasing interest of the pubic towards a complicated geometry and architectures, which are still encouraged by the improving construction techniques and materials of use. The quest to automate the design process had lead the designer to opt for computers to cope with the limited time provided for the design. Algorithms were developed to design problems, but here are aspects of design however which seems to defy algorithmic approach.

One technique that has been around for design and is seriously considered, is optimization technigue. Considering the geometry of the structre ie. Leams, colums, slabs etc. a penalty functon which, if minimioed the optimized solution is arrived at, is idnrtifisd Generally in civil engineering problems cost is the pmaly function, but can be the construction time also.

The constraints which the structre mast satisfy viz. limiting stress, deflection and moment of resistance etc. Which are governed by the code of practice and also the structure design feasibility of the complexity are identified.

There can be many solutions which satisfy the constraints, out of which one has to be opted, this obviously requires experience and precise knowledge of the
area, rigorous numorical computations for each desisn and many such designs should be prepared and finally to get optimum solution satisfying all the constraints.

In the inital stage of design , vialble alternatives are compared and one selected. An inappropriate solution often leads to severe consequences. It is quite natural that the problem is ill defined since the requirements are often finite and the vague factors are to be fixed, depending on the design fulfilling the constraints.

It needs a vast amount of experience and heuristic knowledge for the design. Analytical solutions given by the conventional software do not help much, since a conceptual procedure is desired.

A quite natural problem the designer faces in the design process is the discontinuous constranits, if the argetaints are discontinuous and liable to change with atch haien will obviously need an experienced heuristic knmadge. It is better to have a knowledge base which contains the knowledge of the specified field and a interonce mechanism which can act upon the $K B$ and pick up the relevent rules and infer the best possible conclusions.

The experience and heuristic knowledge is still indispensible and not possesed by all. Thus the need of software which contains the knowledge of an experienced engineer is evitable. If not, a fulfledged but a software which can help, advice and guide the designer to better solution and design.

CBAPTER.3. INTRODOCTION TO EXPERTS SYSTEMS
ES is a computer program that has built into the knowledge and capability that will allow it to operate at the experts level. ES naturally contain large amount of varied knowledge and rules of the specific area and not only the rules and knowledge also but the heuristic knowledge which is the knowledge of the practical experience.

Expert system operate particularly well where the thinking is mostly reasoning not calculating, and that means most of the world knowledge. Even though a lot of professional work seems to be expressed in mathematical formulae, in fact, except in mathematically based sciences, the difficult choices, the matters that set experts apart from beginners, are symbolic and inferential, which are rooted in experimential knowledge. Human experts have acquired their expertise not only from explicit knowledge found in textbooks and lectures, but also from experience by doing things again and again, failing, succeeding, wasting time and effort, then learning to save them, getting a feel for a problem. Learning when to go by the book and when to break the rules. They therefore build up a teritory of working rules of thumb, or "heuristics" that, combined with book knowoledge, make them expert practitioners.

Perhaps the largest single group of expert systems is centered in medicine. The most knowledge-intensive expert system in existence is the INTERNIST CADUCEUS system at the University of Pittsburgh, the creation of a physician, Jack Meyers, and a computer scientist, Harry

Pople. INTERNIST CADUCEUS does diagnoses in internal medicine at a level of expertise that allows it to solve most of the CPCs, or clinical pathological conferences, INTERNIST covers more than $8 \varnothing$ percent of all internal medicine; its knowledge base encompasses about $5 \varnothing \varnothing$ diseases and more than 3,500 manifestations of disease.

Although INTERNIST CADUCEUS was designed to aid skilled internists in complicated medical problems, the program will probably have a future life as a diagnostic aid to physician assistantsd and in rural health clinics, in military medicine, and in space travel.

At Stanford University, several medical expert systems have been designed. MYCIN diagnoses blood and maningitis infections, then advises the physician on antibiotic therapies for treating the infections. Like every other expert system. MYCIN acts as a consultant, having a conversation with its user. the physicia. The physician supplies the patient history and laboratory test results-external data the computer couldn't possibly inferand then the program begins to reason about possible diagnoses. If the physician is uncertain why the program has arrived at a given diagnosis, or why certain drugs have been suggested as therapy; he can ask the program for its line of reasoning:

There is no one specilist who expert ise spans the whole problem. It can only be solved by the interaction of several specilist and the interaction of several specialists and the intelligent fusion of they seperate expertise. Not
always but sometime expert system can manage the intrinsic it is complexity of problems better than human expert. It is this is specifically true of problems that are combinational involving a great deal of trial an error. Trying out combination of problems elements systematically. Problems of design and configuaration examples as are problems of data analysis hypothesis and diagnosis.

DEC engineers use a configuration expert system to plan and manufacture their VAX computers. The systems is reported to plan corruptly in more than 99 percent of the gases. "Prospector" expert system is used by the geologist advising them in exploration of minerals.

## ANOTOMY OF EXPERT SYSTEM

An expert system can contain knowledge-base, inference mechanism, contest, user, interface, and explanation facility and knowledge elicitation facility.


KNOWLEDGE-BASE: The knowledge base contains knowledge specific to the domain of the problem. The knowledge is of two types :
(1) The facts of the domain, a widely shared knowledge, commonly agreed among practitioners that is written in test books and generals of the field.
(2) The heuristic knowledge, which is a knowledge of good practice and good judgement in a field it is basically knowledge of experience whichis acquired over years of practice. This heuristic knowledge gives good guessing which avoids unnecessary such and calculation and leads to the solution. The knowledge base should contain both the knowledge
specified above. Expert system containing facts doesn't mean a fulfledged expert system because expert system is supposed to contain the knowledge of the expert, expert becomes an expert if he can judge or guess correctly witheut trying all possible solutions and picking up the optimum one.

In addition to knowledge, an expert system needs an inference procedure a method of of reasoning used to understand and act upon the combination of krowledge and problem data. The inference procedures, or problem-solving methods, used by knowledge engineers do not need to be arcane or complex,. Even simple methods, used in commonsense reasoning or taught in first course in logic, are adequate. In fact, there is a virtue in employing simple inference procedures since they are easily understood

Now we are able to be more precise about the problem of machine learning; and with this increased precision has come a new term, knowledge acquisition research.

1
This is the most important of the central problens of artificial intelligence research. The reason is simple: the Fower to enhance or amplify the programance of Al programs resides in the specific knowledge of the problem domain that can be brought to bear. Thus, efficient knowledge bases must be large and of high quality.

This knowledge is currently acquired in a very painstaking way, individual computer seientists work with individual experts to explicate the experts heuristics domins those jewels of knowledge out of their heads one by ane. If applied Al is to be important in the decades to come and we believe it must develop more automatic means for what is currently a very tedious, time consuming and expensive procedures. Right now, the problem of knowledge anquisition is the critical bottle neck in artificial intelligence.
updated in terms of its features and properties and theie: relationships with each other in a knowledge base? lheso and other tasks need to be done automatically within the system.

In sum, scientific issues certral to artificial
intelligence underline knowledge engineering and can be enumerated as the parts of an expert system: First it the problem of knowledge representation. How shall the knowledge of a domain of work be represented as data structures in the memory of the computer in a manner in which they can be conveniently accessed for problem solving?

Second is the problem of knowledge utilization. How can this knowledge be used in problem solving? In other words, how should the inference engine be designed?

Third, the most important, is the question of knowledge acquisition. How is it possible to acquire the knowledge so important for problem solving automatically, or at least semiautomatically, in a way in which the computer eases the transfer of expertise from human to the symbolic data struotures that constitute the knowledge representation in the machine? Knowledge acquisition is a long-standing problem of Al. Like 'intelligence' learning has proven to be a eatchall term that's too vague to be useful in creating intelligent computer programs. It served no better purpose to ask whether a machine really could be said to 'learn' than to ask whether a machine really could be said 'think' even when it improved its behaviour by experiance (as one of the earliest programs isn artificial intelligence had done, a program that eventually played championship checkers).
far as possible. This principle is cafled the least commitment principle because variables are not instantiated until more information about the problem space is available Constraint Handling : If the subgoals of the hierarchial planning do not interact with each other, they can be solved independently, However, in practice these subgoals do interact. The interaction between subgoals can be handled by constraint satisfaction method. Constraint satisfaction methods, involve the determination of problem states that satisfy given set of constraints. Essentially this method utilizes constraints to determine the values of parameter in a completely specified problem.

MANAGING THE KNOWLEDGE
For example, one simple form of reasoning that is commonly used is goal-directed backward chaining, the common mental strategy of "working backward" from a desired goal to what you know about achieving it at your starting point.

Al researchers have identified, dissected, and then replicated many such procedures that human beings use all the time, and knowledge engineers, who build expert systems, are skilled at choosing the right set of inference procedures for the type of program they are writing.

An expert. system also requires methods of representing the knowledge it is to contain. This is a technical issue and a matter of some professional dispute, but essentially it means that both a logical structure and a set of appropriate data structured are necessary, through which the special knowledge in the knowledge base can find its way into the memory of a computer.

There is also a formidable problem of knowledge-base management, analogous to data-base management. How shall knowledge be organized, controlled propagated, as well as
and used to find an operator most relevent to reducing this difference．If the operator is not directly applicable to the current situation，then the problem state is changed by setting up sub goals，so that the operator can be applied．需 胞保 an operator has been applied the current state coferponds to a modified state．Means－ends analysis


Problem reduction ：Problem reduction involves factoring problems into smaller subproblems．The problem is represented by means of an AND－OR graph．An AND node consists of arcs pointing to a number of succesor nodes，all of which must be solved for the．AND node to be true．For an OR node，it is sufficient for one of the successor nodes to $\mathcal{T}$ be solved，an OR node indicates that a number of alternative $\omega$ の solutions exist for the problem．In many cases，backward chaining is used to solve the AND－OR graph．

Hierarchial Planning ：The concept of hierarchial planning involves developing a plan at succesive levels of abstraction．In the design of complex systems，the design space is divided into a set of levels where the hither levels are abstractions of details at lower levels，the problem is hierarchially decomposed into loosly coupled subsystems．A number of solutions may exist for each sub system．However，enough information may not be available to ascertain various variables of the subsystem．Further，the solution to one subproblem may depend on the decisions made in the solution of another subsystem．To minimize this dependency，it is important to defer binding decisions as
by end-users, the people being assisted by expert systems. When these users are xeviewing the system's lin! rj reasoning.End-users will not come to trust the reasonjug of an Expert system, and therefore will not use it. Unjess they can easily understand what it is doing.

PROBLEM SOLVING STRATAGIES
Problem solving involves the search for a
solution through a state space by the application of
operators, where the state space consists of an initial
state, a goal state and intermediate states. The solution
consists of all states that lead from the initial
state to goal state.

Forward chaining: A system is said to exibit forward chaining also called as botton-up, data-driven, antecedentdriven, if it works from as initial state of known facts to a goal state. Here. all facts ate input to the system and the system deduces the almost appropriate hypothesis or goal state that fits the facts. The main drawback of his strategy is that it is extremely wasteful to require as input data all the possible facts for all conditions in many gircumstances all possible facts are not knowm or relevent. Gome times the problem solving strategy mechanism is guided by the forwrd chaining is called event-driven. The forward chaining strategy is not appropriate for a design problem if possible goal states of the design problems are not easily represented by a discrete nember of hypothesis. Eackward chaining: A system chaing said to exibit backward
chaining also capled consequent driven, top-down,goal.. driven, if it tries to support agoal stae or hyupothesis by checking known facts in the context. IF the factai the Eintext do not support the hypothesis, then the precinditions that are needed for he hypothesis ar sot up as subgoals. Essentially, the process can be viewed as a search in teh sate space going from toh goal state to the initial staste by the application o inverse operators and involves depthj firat search. he concept of backward chaining may be applied to the decomposition of the tasks in engineering design. If the ucrrent state of the context is not in the proper form, for th completion of a task, the task may be decomposed into subtasks. In this way the ovreall design task may be decomposed into several subtasks and backward chaianing may be applied to each subtask.

Backtracking : The problem reduction approach is applicable to problems that can be subdivided into a tree of fixed subFroblems. However, in a number of practical problems, it may not be possible to decompose problems into a fixed set of problems. A number of alternative approaches may exist. In backtracking, the problem solver backs up to other nodes, at the same level of starting node, if no solution is found along the current path. Backtracking is inbuilt in AI language Prolog.

Means-ends analysis : In means-ends analysis, the difference between the current state and the goal state is determined

## THE SUITABILITY OF PROLOG

The expert system is experted to contain knowledge, inference mechanism and context based decisions are to be taken, avoiding unnessasary search this mechanism cannot be implemented by the conventional languages like BASIC; FORTRAN; PASCAL. In the conventional languages the CPU works its way through a sequence of operations in a predefined way. It can make choices which allow it to follow different paths through the program but only those which the programmer has forseen. This fits well for a numerical problem which are easy to program and can be implemented manually by hand but highly unsuitable to implement and work with knowledge, which is represented by rules.

In conventional languages used for engineering and Scientific work the flow of control is predefined by the programmer. So the programme will be executed in the same sequence of steps. To implement more logical problems using conventional languages calls for a complicated IF THEN ELSE rules and loops, moreover the program is not extendable, additions to the programme is not easy to implement. THE DISADVANTAGES OF THE CONVENTIONAL LANGUAGES

1) The program is executed in a predefined pattern defined by the author:
2) Addition to the program is not easy to implement.
3) Implementation of complex logic calls for complicated loops and IF THEN ELSE rules.
4) The state of variables change with time, so a statement means different depending when it is called.
5) Verifications of the program, to ensure the desired route is taken and the result is achieved is very difficult.
6) Intutive judgement cannot be implemented using conventional languages like BASIC, FORTRAN, PASCAL.

Civil engineering design problems characterised by their inherent imprecision, the paucity and incompleteness of data, and heavy reliance on expert views. Since experience and intutive judgement form and important role in the design process.

A language which can implement heuristic krowledge is needed to implement designing of civil engineering problems. Artificial intelligence languages like LISP, PROLOG are the promising languages to implement knowledge and symbolic computations.

## SUITABILITY OF PROLOG TO BUILD EXPERT SYSTEMS

The PROLOG provides a uniform data structure called TERM, out of which all data as well as Prolog programs are constructed. A PROLOG program consist of a set of clauses where each clause is either a fact about the given information or a rule about how the solution may relate or be infered from a given facts. Thus Prolog can be viewed as first step towards the ultimate goal of "Programming in logic".

Prolog can be viewed as a discriptive language as well as a perspective one. The prolog approach is rather to to prescribe the sequence of steps taken by the computer to
solve the problem. When a computer is programmed in Prolog, the actual way the computer carries out the computations is specified partly by logical semantics of prolog, partly what new facts prolog can infer from the given one and only partly be explicit controlling from information supplied by the programmer.

Imprecision underlines many of the fundamental principles of the subject. Hence our factor of safety, which are quite large because we do not know accurately the magnitude of some of the qaunatities we haye to use.

Expert systems, particularly those writtren in Prolog, may allow ús to incorporate imprecision inte pur design process directly, rather than indirectly. It is not suggested that all design could benefit from this approach, but in certain structural applications where an understanding of the probabilities of failure is required, it would certainly be of benefit.

Imprecision can be incorporated directly into our reasoning in the following way. Instead of the precise statement
$P$ is true if $Q$ is true and $R$ is true
Prolog allows us to write an imprecise version of the same rule as

P is true with support x if $Q$ is true with suport $y$ and $R$ is true with support $z$ and $Y$ combined with $z$ gives $x$

The last clause, which has been deliberately left vague, is used to define some relationship between the support
quantities $x, y$, and $z$. It may be a clause which is specific to the problem in hand, but more likely, it will be a clause written in terms of a standard theory, such as probability or fuzzy logic. Expert systems have already been written in this way, for example the MYCIN project looking at medical diagnosis : and much work is both underway and remains to be done before practical applications can be made in our fields. Not the least of these is the problem of obtaining the basic data for the values of support in the first place.

PROLOG is a logic programming language. It has a sound mathematical' basis, the first order predicate calculus. Wide range of world facts can be represented, by first order predicate calculus, which can be implemented on a computer. The language predicate calculus consists of a number of components such as predicate symbols, variable symbols, function symbols and constant symbols. These features alone give program an edge over the other declarative langiages such as PRIJF and FRIL. Another extremely valid reason for promoting for prolog for expert systems is the expected revolutionary changes to come due to Japanese Fifth generation project (FGCS). The fifth generation languages will be capable of exploring the highly parallel computation architectures to come. This architectures support many central processing units and each CPU simultaneously executes the subdivided problem, which are independent, not sharing the same data or the input is
ready from other CPU for it by the time starts execution.
Prolog . can efficiently support the use of parallel processing.

The inference engine in Prolog is essentially a theorem prover, which tries to prove the goal by proving each of the subgoals starting from the leftmost subgoal in a depth first manner. Hence, the depth first strategy is built into the control mechanism. However, other problem solving strategies can be easily programmed. If any of the subgoals are not satisfied for a particular binding of variables, then the system backtracks; and the program continues with a new set of variable bindings. These variable bindings are available in the database; the variable bindings can also be provided through some user defined functions.

| TOOL/LANGUAGE | DEVELOPER | REPRESENTATION | IMPLEMENTATION |
| :--- | :--- | :--- | :--- |
|  |  |  | SCHEME |
| OPSS |  | RANGUAGE |  |

ADPANTAGES OF DSING PROLOG

1) Prolog is very powerful in manupulating expressions.
2) It has an inbuilt depth first inference engine.
3) Prolog programmes are capable of modelling to some extent the human cognitive ability.
4) Prolog is very dynamic language it can be easily extended:
5) Prolog is easyly readable.
6) Prolog programes are very easy to debug using the built in "Trace" predicate.
7) Recurssion using lists, Prolog can simulate all the functions available in the numerical computation language.
8) Prolog programming can be data directed style, this feature makes prolog dynamic and easy to change.
9) Fuzzy logic and imprecision data can be used in Prolog, Develoment of ES requires flexibility and case of modification. The procedural interpretation of Prolog makes the Prolog program entirely modular. Consider the follow: g set of knowledge-basic rules.
$R$ : if $B$ and $C$ are true. then $A$ is true.
$R$ : if $D$ and $E$ are true. then $B$ is true.
The equivalent Prolog clauses :
$C: A$ if $B$ and $C$
$C$ : B if $D$ and $E$
Note that the Prolog clauses are in fact almost identical to the natural language rules themselves except that the conclusion precedes the conditions. The two rules exist as independent Prolog clauses. Each clause can be mdified or used as an independent procedure. The independent status of clauses makes the Prolog program
entirely modular. Modularity of programs coupled, with the interpretive implementation of most Prolog systems. makes them easy to modify.

ES programs must be able to explain their line of reasoning and Prolog fulfils this requirement by providing an inbuilt inference system a Prolog shell. ESs are develoed incrementally through modification and refinement cycles. This increases the possibility of inconsistencies creeping into the data and knowledge-base. Being a purely logic-based system. Prolog has a much better chance of being able to detect these inconsistencies. Consider the following set of Prolog clauses:

C1: A if not B
C2: B if C
C3: A if C
C4: C
C4 states that $C$ is true from C2 and C3, it there1 :e follows that $A$ and $B$ are true. However, $C 2$ states that $A$ is true only if $B$ is not true. Thus the set of clauses stated are inconsistent, which Prolog will immediately spot. Prolog does not make any distinction between data and knowledge, but providies a uniform formalism for the representation of both the data and knowledge-bases.

## RULES IN THE KNOWLEDGE BASE

The Knowledge base of the expert system contains the knowledge in the forms of rules or in the form of data. This expert system is furnished with the following rules which are of the form of rules and data. The design of the beams strictly follows IS-456. the rules which are relevent and incorporated in the program are listed below.

### 4.2 Effective span

Simply suported Beam : The effective span of a member that is not build integraly with its support shall be taken as clear span plus the effective depth of slab or centre to centre of suppports, whichever is less.

Continuous Beam : In the case of continuous beam, if the width of the support is less than $1 / 12$ of the clear span, shall be as above. If the supports are wider than $1 / 12$ the clear span or $6 \varnothing 0 \mathrm{~mm}$ whichever is less, the effective span shall be taken as under.

1. For the span with one end fixed and the other continuous or for intermediate spans, the effective span shall be the clear span between supports and
2.For end span with one end free and the other continuous, the effective span shall be equal to the clear span plus half the effective depth of the beam or slab or the clear span plus half the width of the discontinuous support, whichever is less.
4.4.1 Arrangement of live loads:
a)Consideration may be limited to combinations of:
1.Design dead loads on all spans with full design live loads
on two adjacent spans, and 2. Design dead loads on all spans with full design live loads on alternate spans.
b) When design live load does not exceed three-fourth of the design dead load the load arrangemets may be design dead load and design live load on all the spans.
22.2 Control of deflection:

The deflection of a structre or part there off shall not adversly affect the appearance or efficiency of the structure or finishes or partitions.

The deflections shall genarally be limited to the following:
a) The final deflection due to all loads including the effects of tempexature, creep and shrinkage and measured from the as-cast level of the supports of floors, roofs and all other horizontal members, should not exceed span/250.
b) The deflection including the effects of temperature, creep and shrinkage occurring after erection of partitions and the application of firishes should not normally exceed span $/ 35 \varnothing$ or $2 \varnothing \mathrm{~mm}$ whichever is less.
4.2.1 For beams and slabs, the vertical deflection limits may generally be assumed to be satisfied provided that the span to depth ratios are not greater than the values obtained as below:
a) Basic values of span to effective depth ratios for spans up to $1 \varnothing$ meters: cantilever 7 Simply supported $2 \emptyset$ Continuous 26
b) For spans above 10 m , the values in (a) may be multiplied by $10 / s p a n$ in metres, except for cantilever in which case deflection calculations should be made.
c) Depending on the area and the type of steel for tension reinforcement, the values in (a) or (b) shall be modified as per Fig. 4.1.


FIG. 4.1
d) Depending on the area of compression reinforcement, the value of span to depth ratio be further modified as per Fig. 4.2.


FIG.4.2.
4.3 Slenderness Limits for Beams to Ensure Lateral Stability A simply supponted or continuous beam shall be proportioned that the clear distance between the lateral restraints does not exceed 60 b or $250 \mathrm{~b} * * 2 / \mathrm{d}$ whichever is less, where 'd' is the effective depth of the beam and $b$ the breadth of the compression face midway between the lateral restraints.

For a cantiliever, the clear distance from the free end of the cantilever to the lateral restraint shall not exceed 25 b or $100 * b * * 2 / \mathrm{d}$ whichever is less.

## REQOIREMENTS GOVERNING REINFORCEMENT.

General: Reinforcing steel of same type and grade shall be used as main reinforcement in a structural member. However simultaneius use of two different types of grades of steel for main and secondary reinforcement respectively is permissible.

Bars may be arranged singly or in pairs in contact, or in groups of three or four bars bundled in contact. Bundles shall not be used in a member without stirrups. Bundled bars shall be tied together to ensure the bars remaining together. Bars larger than 36 mm diameter shall not be bundled, except in columns.

Development of stress in reinforcement :
The calculated tension or compression in any bar at any section shall be developed on each side of the section by an appropiate development length:or anchorage or by a combination thereof.

Development length of bars: The development length Ld is given by $\mathrm{Ld}=\mathrm{Ph} i * S /\left(4 * \mathrm{~T}^{\prime}\right)$
where Phi $=$ The nominal diameter of the bar.
$S=$ the stress in the bar at the section considered
at the design load, and
$T^{\prime}=$ design bond stress given below.
Design bond stress in limit state method for plain bars in tension shall be as below.

Grade of concrete: M15 M2Ø M25 M3 $\quad$ M35 M4


For deformed bars confirming to TS:1786-1979 or IS:1139
1966.

These values shall be increased by $60 \%$ for bars in compresssion the values of bond stress for bars in tension shall be increased by $25 \%$.

Bars bundled in contact: The development length of each bar of bundled bars shall be that for the individual bars, increased by $10 \%$ for two bars in contact, $20 \%$ for three bars in contact, and $33 \%$ for four bars in contact.

Anchoring reinforcing bars in tesion :
a) Deformed bars may be used without end anchorages provided the development length requirement is satisfied. Hooks should normally be provided for plain bars in tesion.
b) Bends and hooks shall confirm to IS-25ø2-1963

1) Bends : The anchorage value of bend shall be taken and four times the diameter of the bar for each 45 degrees of bend subject to a maximum of sixteen times the diameter of the bar.
2) Hooks: The anchorage value of a standard $U$ type hook shall be equal to 16 times the diameter of the bar

Anchoring bars:in compression : The anchorage length of straight bars in compression shall be equal to development length of bars in compression as specified above. The projected length of hooks, bends and straight length beyond bends if provided for a bar in compression, shall be considered for development length.

Anchoring shear reinforcement :
a) Inclined bays: The development shall be as for bars in
tension this length shall be measured a under:

1) In tension zone, from the end of the sloping or inclined portion of the bar and
2) In the compression zone in the mid depth of the beam. b) Stirrups : Not withstanding any of the provision of this standard in case of secondary reinforcemnt such as stirrups and transverse ties complete development length and anchorage shall be deemed to be priovided when the bar is bent to an angle of at least $9 \varnothing$ degrees round a bar of atleast of its own diameter and is continued beyond the end of the curve for a length of atleast 8 diameters or the bar is bent through an angle of 135 degrees and is continued beyond the end of curve for atleast six bar diameters or the bar is bent through an angle of $18 \varnothing$ degrees and is continued beyond the end of curve for a length of atlest four bar diameters .

Bearing stresses at bends : The bearing stress in concrete for bends and hooks described in IS-2502-1963 need not be checked. The beaing stress inside a bend on any other bend shall be calculated as given below.

Bearing stress $=F b t /(R * P h i)$
where Fbt = Tensile force due to design loads in bar or group of bars,
$R=$ Internal radius of the bend and
Phi = size of the bar or, in bundle, the size of bar of equivalent area.

For limit state method of design this stress shall not exceed $1.5 *$ Fck $/(1+2 *$ Phi/a) where Fck is the characteristic strength of concrete and 'a', for a
particular bar or a group of bars shall be taken as the center to centre distance between bar of the group of bars perpendicular to the plane of the bend; for a bar or a group of bars adjecent to the face of the member 'a' shall be taken as the cover plus size of bar (Phi).

If a change in direction in tension or compression reinforcement induces a resultant force acting outward tending to split, the concrete, such force should be taken up by additional links or stirrups. Best tension bar at a reentrant angle should be avoided.

Curtailment of tension reinforcement in flexural members:
For curtailment, reinforcement shall extend beyond the point at which it is no longer requried to resist the flexure for a distance equal to the effective depth of the member or twelve times the bar diameter, whichever is greater except at simple support or end of cantilever.
a) The shear at the cut-off point does not exceed two-thirds than permitted, including the shear strength of web reinforcement provided.
b) Stirrups area in excess of that required for shear and torsion is provided along each terminated bar over a distance from the cut-off point equal to three-fourths the effective depth of the member. The excess stirrup area shall not be less than $\varnothing .4 * b * S / f$, where ' $b$ ' is the breadth of beam,'S' is the spacing and ' $f$ ' is the characteristic strength of reinforcement in $\mathrm{N} / \mathrm{mm}$. The resulting spacing shall not exceed $d / 8$ where 'd'is the ratio of the area of
the bars cut off to the total area of bars at the section, and $d$ is the effective depth.
c) For 36 mm and smaller bars, the continuing bars provide double the area required for flexible at the cut-off point and the shear does not exceed three-fourths that permitted. Positive moment reinforcement
a) At the one-third the positive moment reinforcement in simple members and the one-fourth the positive moment reinforcement in continuous members shall extend along the same face of the member into the support, to a length equal to:
b) When a flexural member is part of the primary lateral load resisting system, the positive reinforcement required to be extended into the support as described in (a) shall be anchored to develop its design stress in tension at the face of the support.
c) At simple supports and at points of inflection, positive moment tension reinforcement shall be limited to a diameter such that computed for by 4.2.1 does not exceed where $m 1=$ moment of resistance of the section assuming all reinforcement at the section to be stressed to: $f=\varnothing .87$ in ther case of limit state design and the permissible stress in the case of working stress design; $V=$ shear force at the section due to design loads; $1=$ sum of the anchorage beyond the centrer of the support and the equivalent at simple support, and at a point of inflection, is limited to the effective depth of the members or 12 , whichever is greater.

Negative moment reinforcemret: At least one-third of the total reinforcement provide for negative moment at the support shall extend beyond the point of inflection for a distance not less than the effective depth of the member or '12' or one-sixteenth of the clear span whichever is greater.

Curtailment of bundled bars-Bars in a bundle shall terminate at different points apart by not less than 40 times the bar diameter except for bundles stopping at a support. Special members-Adequate end anchorage shall be provided for tension reinforcement in flexural members where reinforcement stress is not directly proportional to moment, such as sloped,stressed or tapered footings, brackets, deep beams, and members in which the tension reinforcement is not parali.-l to the compression face.

Reinforcemnet splicing- Where splices are provided in the reinforcement bars, they shall as far as possible be away from the sections of maximum stress and be staggered. It is recommended that splices in structural members should not be at sections where the bending moment is more than $50 \%$ of the moment of the resistance, and not more than half the bars shall be spliced at a section.

Where more than one half of the bars are spliced at a section are where splices are made at points of maximum stress, special precautions shall be taken, such as increasing the length of lap and/or using spirals or closely spaced stirrups around the length of the splice.
4.3 Spacing of reinforcement :
4.3.1 Minimum distance between individual bars :
a) The horizontal distance between two parallel main reinforcement bars shall usually not less than the greatest of the following -

1. The diameter of the bar, if the diameters are equal.
2. The diameter of the larger bar if the diameters are unequal and
3. 5 mm more than the nominal maximum size of coarse aggregate.
b) Greater horizontal distance than the maximum specified in (a) should be provided wherever possible. However when needle vibrators are used, the horizontal distance between the bars of a groove may be reduced to two thirds the nominal maximum size of the force aggregate, provided that sufficient space is left between groves of bars to enable the vibrator to be immersed.
c) Where there are two or more rows of bars, the bars shall be vertically in line and the minimum vertical distance between the bars shall be 15 mm , two thirds the nominal maximum size of aggregate of the maximum size of bar, which ever is geratest.
4.3.2 Minimum distance between bars in tension - Unless the calculation of crack widths shows that a greater space is acceptable, the following rule shall be applied to flexural members in normal internal and exrenal conditions of expose.
a) Beams :The horizontal distance between parallel reinforcement bars or groves, near the tension face of the
beam shall not be greater the value given in the table below depending on the amount of redistribution carried out in analysis and the characteristic strength of the reinforcement.

| Fy | Percentage redistribution |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | $-3 \varnothing$ | -15 | $\emptyset$ | 15 | 30 |
| Clear distance between bars |  |  |  |  |  |
| $\mathrm{N} / \mathrm{mm}^{\wedge} 2$ | mm | mm | mm | mm | mm |
| 256 | 215 | 260 | $3 \varnothing \varnothing$ | 300 | $3 \varnothing \square$ |
| 415 | 125 | 155 | 180 | 210 | 235 |
| 500 | 105 | 138 | 150 | 175 | 195 |
| Table |  |  | 4.4 |  |  |

Cover to reinforcement
4.4.1 Reinforcement shal? have concrete cover and thickness of such cover excluding the plaster and other decorative finish shall be as follows
a) At each end of the reinforcing bar not less than 25 mm , nor less than twice the diameter of such bar.
b) For longitudinal reinforcing in a beam not less than 25 mm nor less than the diameter of such bar.
c) For any other reinforcement not less than 15 mm , nor less than the diameter of such bar
4.4.2 Increase cover thickness may be provided when surface of concrete members are exposed to the action of harmful chemicals, in such increase of cover may be between 15 mm and 50 mm beyond the figures given in 4.4.1. 4.4.2.1 For reinforcement concrete members, periodically immerced in s."
water are subject to sea spray, space the cover of concrete shall be 50 mm more than that specified in 4.4 .1
4.4.3. For concrete of the grade M25 or above the additional thickness of cover specified in 4.4 .2 to 4.4.2.1 may be reduced to half. In all such cases the cover should not exceed 75 mm .
4.5 Requirements of reinforcement for structural members 4.5.1 Beams

Minimum tension reinforcement :The minimum area of tension reinforcement shall not be less than that given by the following

$$
\mathrm{As} / \mathrm{bd}=\varnothing .85 / \mathrm{Fy}
$$

where As $=$ minimum area of tension reinforcement
$b=b r e a d t h$ of the beam
$d=\operatorname{effective~depth}$
Fy $=$ characteristic strength of reinforcement
b) Maximum tension reinforcement : The maximum area of tension reinforcement shall not exceed Ø. $\varnothing 4 * b * D$

Compression reinforcement : The maximum area of compression reinforcement shall not exceed $\varnothing . \varnothing 4 * b$ b . Compression reinforcement in beams shall be enclosed by stirrups for effective lateral restraint.

Side face reinforcements : Where the depth of the web exceeds 750 mm in a beam, side face reinforcement shall be provided along the two faces. The total area of such reinforcement shall be not less than $\varnothing .1 \%$ of the web area and shall be distributed equally on two faces at a spacing
not exceeding $3 \emptyset \varnothing \mathrm{~mm}$ or web thickness whichever is less.
Transvers reinforcement in beams for shear and torsion : The transfer reinforcement in beams shall be taken around the outer most tension and compression bars.

Maximum spacing of shear reinforcement : The maximum spacing of shear reinforcement measured along the axis of the member shall not exceed $\varnothing .75 *$ d for vertical stirrups and 'd' for inclined stirrups at 45 degrees, where 'd' is the effective depth of the section under consideration. In no case shall the spacing exceed $45 \varnothing \mathrm{~mm}$.

Minimum shear reinforcement: The minimum shear reinforcement in the form of stirrups shall be provided such that $\mathrm{Asv} /(\mathrm{b} * \mathrm{~Sv})>=\varnothing .4 / \mathrm{Fy}$
where Asv $=$ total cross-sectional area of stirrips legs effective in shear.
$s v=$ stirrup spacing along the length of the member, $b=b r e a d t h$ of the beam or breadth of the web of flanged beam, and

Fy = characteristic strength of the stirrup reinforcement in $N / m m$ which sahll not be taken greater than $415 \mathrm{~N} / \mathrm{mm} * \mathrm{~mm}$.

However, in members of minor structural importance such as lintels or where the maximum shear stress calculated is less than of the permissible value.

Distribution of torsion reinforcement: When a member is designed for the torsion reinforcement shall be provided as below.
a) The transverse reinforcement for torsion shall be
rectangular closed stirrups placed parpendicular to the axis of the member. The spacing of the stirrups shall not exceed the least of the $X 1,(X 1+Y 1) / 4$ and $3 \varnothing \varnothing \mathrm{~mm}$, where $X 1$ and Y1 are respectively the short and long dimensions of the stirrups.

## THEORY OF DESIGN

The expert system implements the design by 'Ultimate flexural strength' design theory, also called as 'Limit state' of design. The following asssumptions were made in the ultimate flexural strength design. 1. Plane sections normal to the plane of bending remain plane after bending.
2. The strain in concrete at the outermost compression fibre reaches a specified value only at failure.
3. The distribution of compressive stress in concrete at failure is defined by an idealised stress strain curve. 4. The tensile stress in concrete is totally ignored. 5. The stress in the reinforcement is derived from representative stress strain diagram of the steel used.

The first assumption stipulates linear strain profile across the depth, that is the strains in concrete and reinforcement are directly proportional to the distance from neutral axis at which the strain is zero. The ultimate strain in concrete varies between wide limits. The ultimate flexural strength is not appreciably influenced by it. A value of the order of $\varnothing . \varnothing 03-\varnothing . \varnothing \varnothing 35$ is generally adopted for the purposes of design.

The stress-strain relation recommended in the Code is shown in Fig.5.1 with the maximum compressive stress at Ø. 67 Fck (Fck=Fcu) . The compressive stress distribution in the beam is defined as the stress block and can be readily traced as shown in Fig.5.2 in which $X=E c u * D /(Z c u+E 8)$ from similar triangles and 0 in Fig.5.1 corresponds to the
neutral axis.


Fig 5.1
The total compression

$$
C=B * \text { Area } O A B D
$$

and the moment of resistance $M u=C(d-K 2 * X)$
where $K 2 * X$ is the distance of the centroid of the stress block from the outermost compression fibre.


FIG. 5.2
The actual shape of the stress block is not important and in fact the code permits rectangular, parabolic or other shapes to be used which provides reasonable agreement with test results. The equivalence can be easily established. We can deduce the equivalent rectangular stress block for the parabolic shape in Fig.5.3b.

With $d-\varnothing .375 * X=d-\varnothing .5 * a$, we get the hypothetical
depth

$$
\mathrm{a}=\varnothing .75 * \mathrm{X} \text { in Fig.5.3c. Again }
$$

letting $\mathrm{Cp}=\mathrm{Cr}$, we get

$$
2 * \mathrm{~A}^{\prime} * \mathrm{Fck} * \mathrm{~b} * \mathrm{X} / 3=\mathrm{B}^{\prime} * \mathrm{Fck} * \mathrm{~b} * \mathrm{a}
$$ $\mathrm{B}^{\prime}=2 * \mathrm{X} /(3 * \mathrm{a})=8 * \mathrm{X} / 9$.



FIG . 5.3
The depth 'a' is not the depth of the neutral axis. The stress block is very conveniently expressed with the aid of the following three parameters.

$$
\begin{aligned}
& \mathrm{K} 1=\text { Area } \mathrm{OABD} / \text { Area } \mathrm{OEBD} \text { in Fig.5.1, and } \\
& \mathrm{k} 2=\text { Area OABD/Area(x.DB) in Fig.5.2, }
\end{aligned}
$$

The values of K 1 and K 2 for Fig.5.1 are derived in the following:

$$
\begin{aligned}
\text { AreaOABD } & =\text { AreaOAF }+ \text { AreaABDF }=2 * 4 * O D * B D /(3 * 7)+3 * O D * B D / 7 \\
& =17 * A \text { SOEBD } / 21, \\
\mathrm{~K} 1 & =1.7 / 21=\varnothing .8095=\emptyset .81 . \\
\mathrm{K} 2 & =\emptyset .416=\emptyset .42 .
\end{aligned}
$$

The compressive force in Fig.5.2

$$
\mathrm{C}=\mathrm{K} 1 * \mathrm{~K} 3 * \mathrm{Fck} * \mathrm{~b} * \mathrm{X} \quad 2.1
$$

The moment of resistance

$$
\begin{aligned}
\mathrm{Mu} & =\mathrm{C}(\mathrm{~d}-\mathrm{k} 2 * \mathrm{X}) \\
& =\mathrm{K} 1 * \mathrm{k} 3 * \mathrm{Fck} * \mathrm{~b} * \mathrm{~d} * \mathrm{~d}(\mathrm{x} / \mathrm{d})(1-\varnothing .42(\mathrm{x} / \mathrm{d}))
\end{aligned}
$$

The choice of $\mathrm{k} 3=\varnothing .67$ in the Code provides a factor of safety of 3.36 against fallure in concrete as
illustrated in Section 11.2.2.2.
Recommendations of IS-456
The general expressions, 5.1 and 5.2 are readily modified for design by incorporting the partial safety factors Gas and Gmo. The derivations are summarised with reference to Figs 5.4 and 5.5 using $r m s=1.15, \quad r m c=1$.


FIG. 5.4 .
The Code stipulates that at failure the strain in reinforcement should not be less than Es= $\varnothing . \varnothing \varnothing 2+(\varnothing .87 \mathrm{Fy} / \mathrm{Es})$ thereby ensuring a stress of $\varnothing .87 * \mathrm{Fy}$ in Fig. 5.7


Referring to Fig. 5.6, the maximum value $\mathrm{X} 1 \cdot \mathrm{um}=(\mathrm{Xu} / \mathrm{d}) \max =\varnothing . \varnothing \varnothing 35 * \mathrm{Es} /(\varnothing . \varnothing \varnothing 55 * \mathrm{Es}+\varnothing .87 * \mathrm{Fy})$.

$$
=8 ø 5 /(1265+\mathrm{Fy})
$$

The compressive force in concrete

$$
\begin{aligned}
\mathrm{Cc} & =17 * \varnothing .67 * \mathrm{Fck} * \mathrm{~b} * \mathrm{Xu} /(21 * 1.5 \varnothing) \\
& =\varnothing .36 * \text { Feck } * \mathrm{~b} * \mathrm{Xu}
\end{aligned}
$$

The derivations for the moment of resistance of rectangular section is given below.

RECTANGULAR BEAM WITH TENSION REINFORCEMENT
Assigning Asch $=\varnothing$, equilibrium of forces in
Fig. 5.6c


$$
\begin{aligned}
\mathrm{Cc} & =\mathrm{T} \\
\mathrm{X} & =\mathrm{Xu} / \mathrm{d}=\emptyset .87 * \mathrm{Fy} * \mathrm{Ast} /(\emptyset .36 * \mathrm{Fck} * \mathrm{~B} * \mathrm{D}) \\
& =2.417 * \mathrm{P} * \mathrm{Fy} / \mathrm{Fck}
\end{aligned}
$$

where $p=A s t / b d$ and the limiting value of $X^{\prime}$, X'lim is given by eq.5.3. Three possible cases are examined.

Case 1: X' < X'lim
For this case the moment of resistance is conveniently expressed as

```
Mu = Td(1-Ø.42*X')
    = \varnothing.87*Fy*p*b*d*d(1-(\varnothing.42*2.417*p*fy/Fck))
    = Ø.87*Fy*p*b*d*d(1-(1.015 p*fy/Fck))
    =\emptyset.87*Fy*p*b*d*d(1-pfy/Fck)
    =0.87*Fy*Ast(d-(Fy*Ast/Fck*b))
```

> Case $2: \mathrm{X} 1=\mathrm{X} 1 \mathrm{lim}$ $\begin{aligned} \text { Compression } \mathrm{Cc} & =\emptyset .36 * \mathrm{Fck} * \mathrm{~b} * \mathrm{~d} * \mathrm{X} \text { 'lim and } \\ \text { Tension } \quad \mathrm{T} & =\varnothing .87 * \mathrm{Fy} * \mathrm{~b} * \mathrm{~d} * \mathrm{plim}=\mathrm{Cc}\end{aligned}$

The moment of resistance
Mulim $=\varnothing .36 *$ Fck*b*d*d(1-(Ø.42*X'lim)X'lim --5.4 is the limiting value and corresponds to the balanced state in Working Stress Method

Case 3 : x1 > x1 lim
For this case $P>$ Plim and the section should be redesigned for economy implying thereby that increase over Mulim is not permitted for $\mathrm{P}>$ Plim.

RECTANGULAR BEAM WITH COMPRESSION REINFORCEMENT
Such reinforcement would be required when the applied moment 'Mua' on a section is larger than Mulim given by eq.5.4.

The difference is carried by additional tensile reinforcement Ast, compression reinforcement Asc.

Referring to Fig.5.6b, the strain in compression reinforcement $E s c=\varnothing . \varnothing \emptyset 35\left(X u \max -d^{\prime}\right) / X u m a x$ for which let the corresponding stress, read off from the stress-strain curve, be denoted by Fsc.

Asc $=$ Mua - Mu lim /Fsc (d-d1)
Ass $=$ Mua/ø.87*Fy -Mulim $/(\mathrm{d}-\mathrm{d} 1)=$ Asc Fsc $/ \varnothing .87 *$ Fy
The total tensile reinforcement is given by Ast $=$ Plim*b*d + AS2

The presence of Asc is neglected in the computation of Mulim as the modification is not appreciable.

## IMPLEMENTATION OF ANALYSIS IN PASCAL

The implementation of the expert system to design beams and structures is two fold, one is the analysis and the other is the design. First the beam has to be analyzed and the moments produced by the loads are computed which is called the analysis. In the design, the section which can safely handle the loads in the worst circumstances the structure is most likely to be submitted.

All the analysis of the beam is implemerted in the programming language Pascal and the designing, which is more logical and complicated in which the need of the heuristics arise is implemented in Prolog. The designer must opt for only one section, where there can be infinite sections which can satisfy the moments induced. Here the experience and efficiency is demanded to opt for the best of all satisfing the constraints specified.

I confined myself to the problem of analyzing the beams of four types:

1. Cantilever beam
2. Simply supported beam
3. Over hanging beam and
4. Fixed beam

Two types of loadings 1. Point load and 2. Uniformly distributed load were considered. In the analysis of the beams, there can be any number of loads either point load or UDL, and the moments induced by them can be precisely computed. The section of the beam which can safely handle the loads following IS-456 specifications is decided in the designing part of the progaram which is implemented in Prolog.

## ANALYSIS

In the analysis of the beam,it is necessery to compute the moments induced by the loads at every point of length of the beam. So the beam is divided into $1 \varnothing \varnothing$ equal parts which can be represented by an array of dimension $10 \varnothing$ and the corresponding moments and forces are stored in the array. The advantage of using the array is that, the moment at any point on the beam can be accessed dynamically by accessing the corresponding array position at any stage of execution. Instead of using the arrays, packed arrays are used to save the memory space of the computer.

All the computations and variable values are stored and retrieved by using the array $C$ of dimension $1 \varnothing \times 1 \varnothing \varnothing$. This array $C[i, 1]$ where $i$ varies from $\varnothing$ to $1 \varnothing \varnothing$, contains values representing the whole length of the beam. Each segment contains the value of $1 / 1 \varnothing \varnothing$ of length, i.e. if the length of the beam is 12 metres, then $C[\theta, 1]$ is assigned zero, $C[1,1]$ contains 12 and $C[2,1]$ contains 24 and 50 on. So the array $C[i, 1]$ contains the length of the beam in centimeters. Proper care has been taken in the computations to change this units to meters. $C[i, 2], C[i, 3], C[i, 4]$ contain the intermediate values and $C[1,5]$ contains the final bending moments induced by a single load. Many variables, mostly real, strings which read the answers for the queries and flags which are boolean and text files to write the output were used in the program. All the variables are assigned to zero.

The flag OVER which initiates the major while loop
in the program is set to true. This flag is false if the user responds N to the question "Any other loads[y/n] ". The loads are analized one by one and finally added to get the final moments induced. When this ANALYSIS.PAS is called by the DESIGN.PRO, the query "Which type of beam you like to design" appears on the screen. The user is required to enter "cl", "ss", "oh", "fx" to design a cantilever beam, simply supported, over hanging and fixed beams respectively. The responsed of the user is stored in BMTYPE. If the user by mistake opts for none of these, he will be prompted again. It is assumed that the beam is horizontal and all the loads are acting in the downward direction and the supports in the upward direction. This message is displayed on the screen and the user is requested to enter the loads on the beam one after the other. There are two options available point load and UDL. The user is requested to enter " 1 " for point load and " 2 " for the UDL. The first option sets the PTFLAG to TRUE which will later initialize the routines which analyze the point load and the second option sets the UDL flag to true. Now the program splits into two parts, one for calculating the moments and the other for calculating the shear forces for all the four types of beams.

The grade of the steel the user is going to use for the reinforcement has to be specified by the user. If not specified appropriate values are assumed as per IS-456. Three grades of steel FE-5øØ, FE-415 and FE-250 which are HYSDsteel, ribbed tar steel and mild steel respectively. The
user is requested to enter 1,2 or 3 to opt for the steel specified as above. If the grade of steel selected is found unsatisfactory later in the design, the user is requested to change the option simultaneously advising the best suited. If the user still insists on using the same grade of steel, other possible changes satisfying the constraints are deemed. Next the user is requested to enter the grade of concrete he prefers ranging from M1D to M4D. The user can not opt other than these. Next the user is requested to enter the length of the beam in meters, which is assigned to the variable LENGTH. Depending on the option.for the beam type, four separate routines were provided to fix the end conditions. Except in the case of over hanging beam, for all the other three, the support conditions are $x=\varnothing$ and $y=$ length where $x$ is the left support and $y$ is the right support measured from the left. $x$ should never be less than zero and $y$ not greater than the length. A small routine is provided to check these conditions.

If FXflag is true, this flag is assigned true if the user designs a fixed beam, $x$ is assigned zero and $y$ length. It is necessery in case of fixed beam to check the sinking of the supports, which may induce large amount of moments. If any of the supports sink due to construction of foundation inaccuracies, they induce extra moments other than the loads. The user is questioned whether there is any sinking of supports. If the response is ' $N$ ', the analysis of loads starts. If not, a routine which computes the moments induced due to sinking is run.

The user is requested to enter the values of 'E' and 'I' and to specify left or right support which has sunk and also the sinking in centimeters which is assigned to SNK. The moment induced

$$
S 1=6 * E * I * S N K /(L E N G T H * \text { LENGTH) }
$$

At both the ends equal magintude of moment is induced, but in opposite direction. The moment at the support which sinked down is assumed as positive and the other as negative. Since the moment changes linearly from positive to negative, it is easy to implement. The first array segment corresponding to the one end of the beam is assigned the moment computed before i.e S1. The last segment also assigned $S 1$ with the opposite sign. All the intermediate values are computed by linear interpolation and saved in the array H[i].

```
S1 = 6 * E * I * SNK / (LENGTH * LENGTH)
S2 = S1/50; z=S1
FOR i = 1 to 100 DO
    BEGIN
    H[I] := Z;
    Z := Z - S2;
    END
```

The sinking moment is calculated and stored in the
 moment induced by the loads.

The flag ENTRY is set to true which initiates the major loop. All the intermediate values , arrays are assigned zero. The array contains the values calculated for a single load and to arrive at the final moments all of them are added. Immediately after entering the major while entry
loop, assuming at least one load on the beam, the entry falg is set to false. This falg will be assigned true only if tyhe user responds ' $Y$ ' to the question 'any other loads'. Now the user is requested to enter 1 or 2 to opt for point load or UDL. The user is requested to enter the location of the point load which is assigned to 'distance' and the magnitude od the load assigned to ' intensity'.

## CANTILEVER BEAM

A message is printed that the cantilever is assumed to be fixed at the left end. We need such minor restrictions, Otherwise the user has to enter more data. The shear force induced by the load

$$
\begin{aligned}
\text { Sh } & =\text { Intensity } \\
\text { moment } \mathrm{ml} & =\text { distance } * \text { intensity }
\end{aligned}
$$

The moment is zero from the right end upto the location of the point load and linearly increases to m1 upto support. Since the variation is linear it is easy to implement using arrays and interpolation.
$C[i, 5]$ contains the bending moment values and $C[i, 6]$ contains the shear force values.

OVERHANGING BEAM
Simply supported beam is assumed as the over hanging beam with no over hanging spans and a fixed beam is assumed as as simply supported beam with fixed ends, the fixed end moments are separately computed and later super imposed. A single routine developed to analize over hanging beam can be used for simply supported beam as well as fixed
beam. The location of the point load can be

1. In between the supports and
2. On the over hanging span.

In case 1 , no bending moment is induced beyond the supports. The shear force induced in the supports is the magnitude of the force itself but in opposite signs at the two supports. Implementation is quite simple. The value of intensity is stored beginning from the location where the load is located to the support. The same is repeated on the other side with opposite sign.

$$
\text { If } \begin{aligned}
a & =\text { distance between the load and the support } \\
1 & =\text { distance between the supports } \\
b & =1-a \\
i & =\text { magnitude of the load }
\end{aligned}
$$

then the momebt induced just beneath the load

$$
m=i * a * b / l
$$

and the moment at both the supports is zero. The implementation is done by linear interpolation.

UDL
If the user responds with '2', opting for the analysis of the UDL, the user is requested to enter the beginning of the UDL from left and this is assigned to 'distance' and the point wherte the UDL ends and this is assigned to 'span' and the intensity of UDL to 'intensity'. Cantilever with UDL

To analize the cantilever loaded with UDL, let us consider the complicated case of the UDL spanning as shown in the figure above. The shear force is zero between ' $b$ ' and

```
'c' and parabolically increases between 'b' and 'a' and
linearly increasesd between 'a' and support. Numerically
    Shear force at b : \varnothing
    Shear force at a : (a - b) * intensity
    Shear force at support : (a - b) * intensity
    The bending moment
        between b and c : }
        At any point between b}\mathrm{ and a :
            intensity * (b - x) * (b - x ) / 2
        a and support :
        i * ( b - a)/2 + ((a -x)+(b-a)/2)
    where }x\mathrm{ is the point at which moment is
computed,measured from the left.
    The computation of the moment between 'a' and 'b' is
facilitated by FOR loop accessig from 'a' to 'b'.The
coresponding array location to 'a' is assigned
    i * (S2-distance) * (S2-distance)/2 and
```

    S2 is decremented by length/1ø0 each iteration. The
    value computed is assigned to the immediate next array
location representing the moment at that point.

To computet the moment from support to a the same procedure of using a for loop is used. The location corresponding to the support is assigned
i * b - a * (distance between center of UDL and location where moment is computed). The x is decremented by length/ $10 \varnothing$ and this value thus obtained to the immediate next location and so on. To compute the shear force induced in the beam, one part computing between ' $b$ ' and ' $a$ ' and other ' $a$ ' and support. A for loop accessing from $a$ to $b$ is defined as

$$
\begin{array}{rlr}
\text { FOR } i:= & \operatorname{trunc}(r o u n d(d i s t a n c e * 1 \varnothing \varnothing / \text { length })) & \text { to } \\
& \operatorname{trun}(r o u n d(s p a n ~ * 1 \varnothing \varnothing / \text { length })) & \text { do }
\end{array}
$$

Array locations corresponding to distance to span
one after the other. The computation is done in this loop and assigned the corresponding array location systematically.

Simply supported beam : This beam is analyzed as a overhanging beam with no over hanging spans. If the user opts for a simply supported beam, the same routine analyzing overhanging bean is called with the value of $x$ and $y$ assigned to $\varnothing$ and distance.

Over hanging beam : The analysis of over hanging beam with UDL is the most complicated case of all the cases. To compute the shear force, the problem is subdivided into three parts for clarity.

```
case1 : Span <= y
```



FIG
The first step is to calculate the reactions of the supports ' $b$ ' and ' $c$ '. In this case of 'span $<=y$ ' the moments about 'c' were computed. For example, considering the moments about ' $c$ '

$$
\begin{aligned}
& i *(c-a) *(c-a) / 2+b *(c-b)=\varnothing \\
& b=i *(c-a) *(c-a) / 2 /(c-b)=R 1 \\
& R 2=(\text { Intensity } *(C-A))-R 1
\end{aligned}
$$

To compute the shear force over the nbeam 3 arrays C[I,6], C[i,7], C[i,8] were reserved, $C[i, 7], C[i, 8]$ storing
the intermediate vriables and $\mathrm{C}[\mathrm{i}, 6]$ containing the final shear force values. The shear induced by another load is added to this $C[i, 6]$ in each iteration the analysis, finally at the termination of this analysis routine this array contains the total shear force values induced by all the loads.

```
if (span <= y) then
        begin
            R1 : = ( (span-distance) *intensity *.
            ( \(y\)-distance-(span-distance)/2) )/(y-x);
            zs := (span-distance)*intensity /
                            (1øø/(length/(span-distance)));
            zh1:=ø; g1:=g1+r1;
            for \(i:=\) trunc(round(distance*1 \(1 \varnothing \varnothing /\) length)) to
                    trunc (round (span*1 \(\varnothing \varnothing /\) length)) do
                    begin
```

                        \(c[i, 7]:=z h 1 ;\)
                        zh1: \(=c[i, 7]+z s ;\)
                end;
            for \(B:=\operatorname{trunc}(\) round \((\operatorname{span} * 1 \varnothing \varnothing /\) length \())+1\) to \(1 \varnothing \varnothing\) do
                    \(c[i, 7]:=z h 1 ;\)
        end
    Here $Z S$ is the loading on the . $\varnothing 1$ of the beam span. A FOR loop accessing from 'distance' to 'span' assigns ' $Z S$ ' to the location corresponding to the 'distance' and increment that by ' ZS' and assigns to the immediate next location and so on upto the location accessing the 'span', this simulates the loading over the span. The remaiining span is assigned the maximum value that is (Distance-Span)*intensity.

Another FOR loop from ' $b$ ' to ' $d$ ' assigns R1 to all the array location, and another FOR loop adds $R 2$ to the array from $C$ to $D$, So now $C[i, 8]$ arry corresponding from 'b' to ' $c$ ' contains the $R 1$ and from ' $c$ ' to ' $d$ ' contains $R 1+R 2$, this is exactly simulates the shear resitane of the
supports. This values when subtracted from the shear induced by the loads gives the resultant shear force induced in which we are intrested. C[i,7] is cantaining the shear induced by the loads, So $C[i, 7]-C[i, 8]$ gives the resultant shear induced.

The instruction
$C[i, 6]:=C[i, 6]+C[i, 7]-C[i, 8]$
assures the total shear induced by all the loads.


FIG
In this case the UDL is placed over the right over hanging span of the beam. This case needs a seperate routine only to compute the reactions, if reactions at both the supports are obtained the same routine above is used to compute the shear.

To compute the reaction, moments about the support C are cosidered

```
    R1=((Span-distance)*intensity)-(Span-Y-(Span - Distance)/2)
    R2=(Span - Distance)* intensity - R1
```

Case 3 : ( Distance <= Y ) and ( Span >= Y )
In this case the udl starts from a point before the support and spans beyond the support as shown in the


The reaction of the left support $R 1$ and the reaction of the right support R 2 are computed as below.

$$
\begin{aligned}
& \mathrm{R} 1=\left(\text { Intensity* }\left((Y-\text { Distance })^{\wedge} 2\right) / 2 /(Y-X)-\right. \\
& \text { ((Span-Y)^2)*intensity/2/(Y-X) } \\
& \mathrm{R} 2=\text { (Span-distance)*intensity -R1 } \\
& \text { The same routine as in Casel is used to compute }
\end{aligned}
$$ the shear forces.

At this stage, it is neccesary to check the direction of the supports assumed before are correct or not, It is assumed that the loads are acting downwards and the the supports are in upward direction, but in the case of overhanging beams if the loads act over the overhahging span the far support should be in the downwards direction, as shown below.


FIG
This can be very easyly checked by using the R1 and. R2 vlaues. Since R1 and R2 are computed asssuming that
the direction of the supports is upwards, and R1 and R2 must be a positive values if the direction assumed is correct, but if the R1 is negetive means the left supprt should be downwards, else if $R 2$ is negetive $R 2$ must act downwards.

The user is intimated about this direction changes and proceeded.

## Bending moment

A single routine is developed to compute the moment of the overhanging beam, this is simple since the reaction of the supports are computed already. The moment at any point $X$ on the beam be arrived at by taking the net of the moments acting at the point. Arrays $C[i, 2]$ and $C[i, 4]$ were used solely for this computatioins. For convienience $M, N$ are defined, these are the integers which correspond to
'Distance' and 'Span' array locations.

```
L1 := 1\varnothing\varnothing * distance/length ;
L2 := span * 1øø /length;
M := trunc(round(l1)) ; N := trunc(round(l2));
    z:= \varnothing;
    for i:= m to n do
        begin
            c[i,2] := z;
            z := (z+intensity *length/10\varnothing);
            end;
        for i:= n+1 to 1\varnothing\varnothing do
            c[i,2] :=intensity *(span-distance);
        for i := \varnothing to 1\varnothing\varnothing do
            c[i,4]:= - (c[i,2]*
                                    (C[I,1]/10\varnothing-distance)/2 );
        for i := trunc(round(100*x/length))
                            to trunc(round(y*1ø\varnothing/length)) do
                begin
                    if c[i,1]< span*1\varnothing\varnothing then
                    c[i,4] := r1 *(c[i,1]/1\varnothing\varnothing-x)
                    -c[i,2]*(c[i,1]/10\varnothing-distance)/ 2
            else if c[i,1] >= span*1\emptyset\emptyset then
                    c[i,4]:= r1 *(c[i,1]/1\varnothing\varnothing-x)
            -c[i,2] * (c[i,1]/100-distance
            -(span-distance)/2 );
```

For the fixed beam extra fixed end moments are computed, considering every load and finally adding to get the total moment. THe net moment is arrived at by subtracting this end moments from the moments induced by the loads, That is the advanage of using a fixed beam than a simply supported beam, because net moment induced in the fixed beam is less.

To compute the fixed end moments due to a UDL 'Area moment method' is used. According to Mohr's theorem "The area of the free and fixed bending moment are numerically equal or the resultant area of the bending moment is zero".

This leads to the equation

$$
\begin{aligned}
& A^{\prime}=(M 1-M 2) / 2+L \\
& A^{\prime} X^{\prime},=L^{\wedge} 2(M 1+2 * M 2) / 6 \\
& \text { where } M 1, M 2: \text { moments at the ends } \\
& L \text { length of the beam } \\
& A^{\prime} \text { area of the Fixed momemt diagram } \\
& X^{\prime}, \text { distance of } C . G .
\end{aligned}
$$

Substituting the other values and Solving both this equations the values of $M 1$ and $M 2$ are arrived at.

M1 is assigned to the left most point and $M 2$ to the right most point and the irtermediate values are
interpolated leniarly, as

```
for i := Ø to 99 do
    begin
                z1:=z;
                z:=z+(c[i,5]+c[i+1,5])/2
                    *(length/1ø\varnothing);
    end;
uarea:=z1;
```

This routine above compues the total area of the bending moment induced by the loads, and this is assigned to 'UAREA', this correspond to A in the equatons above.

$$
z:=\varnothing ;
$$

for $i:=\varnothing$ to 99 do
begin
$z:=z+(c[i, 4]+c[i+1,4]) / 2 * l e n g t h / 1 \varnothing \varnothing *$ (c[i,1]/1ø $\quad$ length/2øD);
z1:=z;
end;
umax: =z1;
The routine above computes the moment area of the bending moment diagram, this assigned to 'UMAX', corresponding to A'X''.

The fixed end moments are computed bt the routine.
$z 1:=4 * u a r e a / l e n g t h-u m a x * 6 / l e n g t h / l e n g t h ;$
z2 := 2*uarea/length-z1;
$\mathrm{m} 1:=\mathrm{m} 1+\mathrm{z} 1$;
$m 2:=m 2+z 2$;
The intermediate values are interpolated and assignd to the corresponding locations, and assigned to G[i] and the net bending moment to $F[i]$ as

```
z := m1;
for i:= Ø to 1\varnothing\varnothing do
    begin
        g[i]:=2;
        z:=z+(m2-m1)/100;
        f1:=g[i]-c[i,5];
        f[i]:=-(f[i]+f1);
    end;
```

After analysis of loads the next step is to display the analysis by a graphic routine. Infact the bending moment diagrams are better appreciated by the graphical display. This is explained clearly later.


## LINKING PRCLOG AND PASCAI

The expert system ta arialyze ard desigri beams should be capable of performing both the complex rumerical computatiors and also the symbalic computations workirg with the krowledge base. It is not an easy task to implement both rumerical and symbalic computaticns using a single programming larguage like Pascal or Prolag. If orly Pascal is used, the kriowledge arid rules may be simulated arod implemented but extendirg the programs adding few more rules is very difficult. If only prolog is used, it is easy ta implemerit symbolic computatiors, but the simulatior of sybolic cxomputation like arrays by lists ard recursion makes the program very complicated arid urireadable. The only passible salution is to use both the symbolic computation iarguage and rumerical computation iarguage ard build ari effective lirik betweer them to trarisfer data. The lirking car bue dorie in mariy ways, but here files were used to trarisfer data from Pascal to pralag. The pascal program "analysis. pas' is callec ir pralag erivirorment ard executed. 'Arialysis.pas' reads the iriput of beams arid loads arid arialyzes them and writes the data in a text file 'data. dat' ard displays graphically the aralysis ard termiriates. Later this file is opened ir Pralog envimonment and the data is read from "Data.dat'. Prolog has an iribuilt predicate "system("DOS commarid")", this predicate transfers contral to the operating system and executes the DOS commard giver in the quotes and regains the control. Turbo Pascal has a provision to create an executable file, the executable file
'arialysis.com' of 'analysis. pas' is created ard the predicate 'system("arialysis.com")" executes the file "aralysis.com', which arialyzes the beam ars creates the file 'data.dat' iri Prolag erivirormerit.

The disadvantage of this technique of trarsferirig data through $I / 0$ files is relatively slow compared to transfer of data by linking of routines, but still the data trarsfer thrcugh files is prefered due to it's simplicity ard flexibility Moreover the data trarsfer is performed orily once. The data read from the 'data. dat' is stared irn the knowledge base which cari be referred at any stage of executiar.

The grade of cocrete, the grade of steel arid the diameter of the bars, the type of beam ard ladirg specificaticrs weme read by irteraction iri "aralysis. pas" and the loads are aralyzed and all this data is writter in data. dat. The grade of the coricrete ard the grade of steel specified by the user are checked for the safe desigr. If they are fourid to be unsafe, and chariges ame rieeded. This is intimated to the user and requested to charge the specifictians. If still the user insists in using the same grades, other alternatives are corsidered. A small routirie is used to pick up the maximum and mirimum shear force arid bending moment values and the momerts at different points were picked up arid alcong with other data were written in "data.dat'.

The amalysis is better appreciated by the user if the output is displayed graphically. Obvigusly the variations of the momerit arid shear should be displayed graphically rather thar rumerically. The graphics routire is called if the usser warits the output graphically. If the bean laaded with heavy laads, the momert values are high arid low if laaded with relatively low iriterisity of laads. The beriding momert diagram should not be so small if the laads are of low magritude beyond the comprehension of the humari eye arid should not be very large abriomally also. So a relative diagram is displayed. The maximum value of beridirg momerit is picked ard 40 pixels were allated to represert that vlaue ard all orther values are displayed relatively. Sc a beam ¿aded with UDL it's full length with $1.0 . \mathrm{T} / \mathrm{m}, 10 . \mathrm{T} / \mathrm{m}, 100 . \mathrm{T} / \mathrm{m}$ will display exaxctly the same berding moment diagram. The
actual implemertation is
tegirt
graphcolarmode;
$r:=0$;
for $i:=0$ to 100 do
if $a b s(c[i, 5])\} r$ then $r:=a b s(c[i, 5])$;
if $r\{<0$ then
begirs
$\mathrm{J}:=48$;
draw $(50,50,248,50,1) ;$
for $i:=1$ to 100 da
begin
$1:=$ truric (rourid \{c[i,5]/ $r * 40.0)$ );
$\mathrm{J}:=\mathrm{J}+$ Е;
$k:=50-1$;
draw ( $J, 50, ~ J, k, 2)$
end;

In case of fired beam, first the moment diagram is displayed and over it, the separately computed fixed erd moments are super imposed ard finally below it, the resultant

```
af both the moments is disfalyed. A single routine is used to
display the brn diagram for all the four types of beams.
    if frflag=true ther,
        begin
        j:=49;
        draw (50, 30, 250, 30, 1);
        for i:= 0 to 100 do
            begin
                    1:= trunc(rourid (g[i]/r * 40.0));
                    j:=3+E;
                    k:=50-1 ;
                    draw (J,50, J, k, 3);
            erid;
        J:=47;
            for i:= 0 to 100 do
                    begir,
                begir,
                        l:= truric (raurid{f[i]/ r * 40.0));
                        \jmath:=3+2;
                        k:=30-1 ;
                        draw(J, 70, J,k,E);
                erid;
                r:=0;
        for i:= 1 to 100 do
            if abs(c[i,6]) > r then r:=abs(c[i,6]);
                if r<<0}\mathrm{ ther,
                        begin
                                    J:=48;
                                    draw (50,160,248,i60,1);
                                    for i:= 1 to 100 do
                                    begin
                                    l:= truric(round (c[i,6]/r * 40.0));
                                    \jmath:=\jmath+こ;
                                    k:=160-1 ;
                                    draw (J, 160, J, k,2)
                                    end;
                                end;
```


## DESIGN IMPLEMENTATION IN PROLOG

The design part of the program has been implemented om Prolog. This section clearly explains the techniques used for the implementation. Prolog programs are basically a declarative. The decleration were made in the 'Domains',' Predicates', 'Clauses'.

Domains : "mf" is declared as a file and later assigned to data.dat and opened to read.

Data base : data base contains the facts which are computed and generated during the process of execution. The computed values which are needed many times during the execution are also stored as facts. This technique of storing the variables as facts may increase the 'search' in the program. But transfering many variables from one predicate to the other makes the program look complicated. In the progarms which are relatively small, it is better to store the variables in the knowledge base and later extracted. The beam(string), steel(integer), stldia(real), length(real), pbmd(real). The length of the beam is extracted from length( $X$ ), $X$ is bounded to the real value of length(real). The safe breadth and depth computed as per IS456 are stored as safe_breadth(real), safe_depth(real) and the final breadth and depth which is decided later by interaction as a final_breadth(real) and final_depth(real).

Predicates : The predicate section is declared with numerous predicates each contributing a solution for the small divided problem of the major task of designing the beam. The flow of control and data is described clearly in
the flow-chart.
Start : The predicate start is the first and at the top of the hierarchy. In fact, the goal of the program is to prove the predicate 'start' true. This predicate triggers all other predicates. This predicate tries to prove 'True' all the predicates.

```
    " start :- write("Letsstart"), openread(mf,"data.dat"),
readdevice(mf),readln(Beam), readreal(LENGTH), readln(CC),
readint(SS), readreal(NB), Nbmd=1.5*Nb, readreal(PB), \(\mathrm{Pbmd}=\mathrm{Pb} * 1.5, \quad\) readreal (Sh1), \(\quad \mathrm{S} 1=\mathrm{Sh} 1 * 1.5\), readreal(Sh2), S2 \(=\) Sh \(2 * 1.5\), readreal(Stldia), readreal(Depth), readreal(Breadth), closefile(mf), asserta(beam(Beam)),
``` asserta(concrete(CC)), asserta(steel(SS)), asserta(nshr(Sh1)), asserta( pshr ( Sh2 ) ), asserta( length ( LENGTH ) ), asserta ( nbmd ( NBMD ) ), asserta ( pbmd ( PBMD ) ), asserta( pascald( Depth) ), asserta ( pascalb( Breadth )), asserta(stldia(Stldia)), window, bmfactor, !, conc, !, stl, cvr(Stldia), safe_section(Length), option, breadth, design. This predicate first tries to prove true a write statement and prints the message of introduction. The next predicate is 'system("analysis.com")'. This is a very important predicate as explained before which links Pascal and Prolog. By executing the single predicate, all the task of reading the input and loads analyzing the loads and displaying it graphically and creating a file of data. The next task is to open the data file 'data.dat' created by 'analysis.com' and read the data and store them in the data base.

The values witten in the file were read and stored in the knowledge base by using the predicate "asserta(length(Length))", this predicate asserts the characters in the quotes as a clause to the program. The shear force and bending moment values were multiplied by 1.5 the partial safety factor to correspond to the limit state of design. The next predicate creates a window with the header 'Auto beam" in reverse video. All the interactions and input out put is through this window. The predicate 'bmfactor' decides a factor which is the maximum ratio of length and depth of the beam section. This gives the minimum depth to be provided for the safety against the deflection of the beam under loading. IS-456 clearly specfies the minimum ratios of length and depth to be provided for safety. This factor is computed by this predicate and assertas in the database 'bmfac(Factor)'. The next predicate to be satisfied is 'conc', this predicate asserts ther flexural strength of concrete given the grade of concrete, and the precdicate 'stl', asserts 'stlgrd(Grade)' and also asserts a variable xlim(X) which is derived by the grade of steel and needed later for computations. The 'cover' predicate decides the minimum cover to be provide depending on the diameter of the bars to be used for reinforcement. This diameter is specifed by the user, if not it is arrived at by knowing the length and the depth is estimated, from which the diameter is assumed. The next predicate triggered is the 'safe_section' predicate,
this decdes the safe section dimensions for the beam as per IS-456. The predicate 'option' writes the message that the expert system designs a singly reinforced section if the user does not specify the dimensions of the beam. This is necessary because the user may specify the loading, the grade of steel and concrete he likes to use and lever other details to the system itself then the user must be informed that the beam going to be designed is a singly reinforced but not doubly reinforced.

Breadth : The predicate breadth decides the section of the beam. There can be infinite sections which can safely handle the loads and moments thereby induced, but only one section can be opted for. The program displays six possible sections which can safely handle the loads with different depth breadth ratios varying from 1.5 to 4 . This acts as a guidance to the user if he wants to specify the section. Next the user is questioned if he wants to specify depth/ breadth or both. If the user specifies the depth,the breadth is decided else if breadth specified, depth is decided and if both the dimensions were specified by the user, the validity and safety of the section is checked. If the section specified is proved unsafe, other changes in the design are made. This predicates breadth displays this information and calls decide_depth predicate.

The decide_depth predicate computes the sections with depth/breadth ratios varying from 1.5 to 4 and displays them and triggers 'dcd'. The 'dcd' predicate reads the
response of ther user and calls 'dod1(Reply)'.
dcd1(N): A message is prited that the design is based on IS-456 specifications and a single reinforcement beam is going to be designed and triggers 'prélim_design'.
dcd1(Y): This predicate reads the breadth or depth the user wants to specify and calls decide_b(Depth) if depth is specified or decide_d(Breadth) if breadth is specified.
decide_b(D) : This predicate has to decide the breadth. The predicates length(L), \(x\) _lim(X) etc. when tried to prove true, prolog searches the knowledge base and binds 'L' and ' \(X\) '. This is how we can regain the variable values from the knowledge base. The breadth is arrived by substituting other values in the equation.
\[
M=\varnothing .36 * \mathrm{Fck} * \mathrm{~B} * \mathrm{D} * \mathrm{D}(1-\emptyset .42 * \mathrm{X}) * \mathrm{X}
\]
where \(B\) and \(D\) are breadth and depth of the beam. The breadth computed by this equation is rounded and a clear cut data of the situation is presented to the user by printing the values of the design moment ' \(M\) ', depth specified by the user ' D', bredth which can safely handle the loads ' \(B\) ' and the minimum breadth to be provided as per IS-456. The greater of both the breadths is selected for design and the user is questioned " Is the section satisfactory?". The response of the user is bound to the Reply and 'hm1(Reply)' is triggered.
decide_d(B) : This predicate decides the depth of the beam section io the user specifies the breadth and depth to be decide by the system. This predcate as 'decide_b' not only computes the other dimension but also gives the minimum
depth should be povided by considering the modification factors into account. The depth provisions in IS-456 can be further modified considering the modification factors which makes the design more economical. The predicate 'modfac' is called, which approximately computes the percentage of steel in the section and reads the corresponding value for the grade of steel from the data base as below.
```

modfac_tsn(\varnothing.\varnothing,2.1\varnothing\varnothing,2.4\varnothing\varnothing, 3.\varnothing\varnothing\varnothing).
modfac_tsn(\varnothing.2,1.350,1.6\emptyset\emptyset,2.30\varnothing).
modfac_tsn(Ø.4,1.Ø75,1.28Ø, 2.1Ø\varnothing).
modfac_tsn(\varnothing.6,\varnothing.95\emptyset,1.1Ø\varnothing,1.73\emptyset).
modfac_tsn(\emptyset.8,\varnothing.90\varnothing,1.\varnothing25,1.525).
modfac_tsn(1.\emptyset,\varnothing.85\emptyset,\emptyset.95\emptyset,1.415).
modfac_tsn(1.2,0.825,0.925,1.320).
modfac_tsn(1.4,\varnothing.81\varnothing,\varnothing.9\varnothing\varnothing,1.285).
modfac_tsn(1.6,\varnothing.8\varnothing\varnothing,\varnothing.88\varnothing,1.215).
modfac_tsn(1.8,\varnothing.79\varnothing,\varnothing.860,1.15\emptyset).
modfac_tsn(2.\varnothing,\varnothing.775,\varnothing.84\varnothing,1.125).
modfac_tsn(2.2,\emptyset.76\varnothing,\emptyset.82\emptyset,1.1\varnothing\emptyset).
modfac_tsn(2.4,\varnothing.745,\varnothing.8\varnothing\varnothing,1.\emptyset85).
modfac_tsn(2.6,\varnothing.73\emptyset,\emptyset.785,1.\emptyset65).
modfac_tsn(2.8,\varnothing.715,\varnothing.770,1.045).
modfac_tsn(3.\varnothing,\varnothing.7\varnothing\varnothing,\varnothing.755,1.Ø25).

```

Knowing the grade of steel and the percentage of
steel the corresponding ratio is computed by interpolation of the two nearest values. After the displ:y of the section if the user in satisfied the predicate gm1, gm2 are called as explained in the predicate 'decide_b'.
hm1 (Reply) : If the Reply is 'y', the user is thanked for considering the advice of the system and proceeded to design. Since the breadth and depth are decided. The reply is ' \(N\) ', the message to specify the breadth or depth have to be changed. The response is bound to \(A\) and \(h m 2(A)\) is called.
requested to enter the depth he prefers which is bound to ' \(D\) '. At this stage, the user wants to change the depth he specified first. So once again the predicate 'decide_b(D)' is called else if \(A\) is ' \(B\) ', the breadth is read and bound to 'B'. At this stage, the user first specified the depth and later specified the breadth considering the data displayed. So the expert system is provided with both the dimensions. The next step is to chech the validity of that section. Perilim_design(B,D) is called.
prelim_design( \(B, D)\) : This predicate decides if the breaths and depth are capable of handling the loads specified by a singlly reinforced section. If not safe, the user is questioned if the design can be for a doubly reinforced beam. iF the rersponse is 'Y', 'double(B,D)' is called.If the response is ' \(N\) ' the predicate, means the user does not want the beam to be design a doubly reinforced. So the program is backtracked to the predicate 'breadth', once agin teh user is facilitated to chage the the dimensions of the section.

The user after deciding the section, and the sectioon is proved safe satisfying all the codal specifications the predicate 'design' is triggered.This predicate designs the final design and the percentage of steel and the steel details are fixed \(y\) the predecate 'stl_cal'. The knowledge base is furnished with the safe and minimum percentages to be satisfied as below.
```

max_prsnt_rnf_sngl(15,250,1.32).
max_prsnt_rnf_sngl(15,415,\varnothing.72).

```
```

max_prsnt_rnf_sngl(15,5Ø\varnothing,\emptyset.57).
max_prsnt_rnf_sngl(20,250,1.76).
max_prsnt_rnf_sngl(2\emptyset,415,\varnothing.96).
max_prsnt_rnf_sngl(20,50\emptyset,0.76).
max_prsnt_rnf_sngl(25,250,2.20).
max_prsnt_rnf_sngl(25,415,1.19).
max_prsnt_rnf_sngl(25,5Ø\varnothing,\varnothing.94).
max_prsnt_rnf_sngl(30,25\emptyset,2.64).
max_prsnt_rnf_sngl(30,415,1.43).
max_prsnt_rnf_sngl(30,50\emptyset,1.13).
values of the reinforcement.

```
```

OROGFAM Arrlysis(imput, output);
UAR

```

```

    end;
    procedure cal;
    begirs
                p:=0;
                for i:= 0 to 100 do
                begin
                    c[i,1] := p;
                            p:=p+length;
                end;
            p:=0;
            for i:= 0 to 100 dc
                        begin
                                    c[i,Z] := p;
                                    p := ddud]*c[i,1]*口[i,1]/20000.0;
                    end;
                z:=ddudl*lerigth/E;
                p:=0;
                for i:=0 to 100 dc
                        begin
                        c[i,9] := (2*с[i,1]/100) - (c[i,Z]);
                    erid;
        end;
    begir,
clrscr;
over := true;
while over=true do.
begin
qver := false;
for I:=0 to 100 do
begir,
c[i,1]:=0;
C[i,こ]:=0;
c[i, 3]:=0;
c[i,4]:=0;
c[i,5]:=0;
end;
clflag:=false;ssflag:= false;
ohflag:= false;fxflag:=false;
ptflag:=false;udlflag:=false;
flag := true;
writelri;
writelr ("Which type of beam you like to desigri);
while flag= true do
begin
flag:= false;
writelr;
writelri"The follawing options are avilable');
writelr("Caritilever beam [cl]");
writelri("Simply supported beam [ss]');
writeln('Over hargirig beam [oh]');

```
```

writelr,('Fixed beam
[Fx]5);
writelri;
write("Eriter your selected aption by entering the');
write(' characters iri the braces : ');
readlri (ariswere); tmtyoe :=ariswere;
if (( ariswere ='ss') or (ariswere ='5s'))
ther, ssflag:= true
else if({ariswere='ah')ar(ariswere='0H'))
ther, chflag:= true
else if ({ariswere='fx')gm(ariswere='FX'))
then fxflag:= true
else if {ariswere='c1') or (answere='CL')
then clflag:=true
else
begir
write(" Irvalid optiorr ');
write('please try again');
writelri
flag:=true;
erid;

```
    end;
flag:= true;
    while flag \(=\) true do
        begir
            flag: \(=\) false;
            cirsce:
            write(" Which grade of steel');
            wtitelri" is used for reirifircement');
            writelr;
            writelri" The failawing optiors are available");
            writelri"Type " 1 " for FE 500 grade steei);
            write "Type " \(z\) " for FE 415 gracie steel --");
            write (" Ribbed Tar");
            writein("Type " \(3 "\) for FE 250 grade steel");
            writelri
            write ( What is yolir aptiori \(:\) ' ; read (i);
            if \(\{i=1 ;\) Gr \(\{i=\bar{E}\}\) or \((i=E)\)
                ther, orade: \(=\mathrm{i}\)
                else
                    begir
                    writelri(' Sarry irivalid aption try again');
                    writeir:
                        flag := true:
                    erid;
            writelri;
            write('Specify the diameter af the steel");
            write\{' bars [y/n] : '); readln(answere);
            if (ariswere='y') or (ariswere=' \(y^{\prime \prime}\) ) thers
                begin
                    flag:=true;
                    while flag=true
                        begirt
                    flag:=false;
                    write('Eriter the diameter of bars :');
```

                readlr(stldia);
                        if stidia=0 ther, flag:=true;
            erid;
        end;
        if (ariswere='r') or (ariswere='N') ther,
            begir,
                sdflag:=true;
                stldia:=1.8;
            end;
    erid;
    flag:= true;
while flag = true do
begir
flag:= false;
clrser;
writelri;
writelri("Which grade af coricrete you prefer ');
writelrig
WRITELN("The following optiars are available");
writeln{"M10,M15,MEO,MES,M30,M35,M40, ) ;
writelri;
write{'Eriter your selected aptiors:');
readiri(reply);
if (reply='m10') ar (reply = 'M10') ar
(reply = 'm\5; ar (reply = "M15') ar
(repily ='mEO') Gor (repiy = "MEO") or
(repily ='mes') or (repiy = 'MES') or
(reply = 'n30') ar (repiy = "MSO") or
(reply ='mS5') ar (reply ='MZ5') or
(repiy = 'm40') ar (reply = 'M40')
ther, caricgrad:=reply
else
begir,
writelri(" Sorry imvalid optiaritry agairi');
writelri;
flag := true;
end;
erid:
clrser;
writeln;
writelri("The suppports are assumed to act ");
writelri ("ir, the upward direction"); ;
writeln("and the laads in the dawnward direction');
writelri;
write('Please enter the length of the beam (mts) :');
readln(length);
if (bmtype ='cl") ar (bmtype='Cl') ther, depth := lerigth/7 ;
if (bmtype ='ss') or (bmtype='SS') thers depth := length/20;
if (bmtype ='ah') or (bmtype='OH') then depth := length/己6;
if (butype ='fx') or (bmtype='FX') ther depth := lerigth/26;
{ if depth-truric(depth)>0.5 theri depth := trunc(depth)+0.5;
if depth-truric(depth) {0.5 ther, depth := truric{depth);}
breadth:=depth/E;
writelr("The expert opinion is to have');

```
```

writelri("Maximuri depth [rnts] : ", depth);
writelri("Maximum breadth[mts]: ",breadth);
write\'This approximate depth ard breadth satisfactory : ');
readln{ariswere);
if (arswere=' }n\mathrm{ ') or (answere='N') then
begin
write(" Do you have ary commitments regarding");
write{" the breadth or depth of the beam[y/r]');
readlri(answere);
if (ariswere ='y') or (arswere='Y') theri
begir
write('Ereadth ar Depth [b/d]');
readlri(reply);
if (reply = 'b') or (reply='E') then
begir,
write('Enter the breadth af the beam specified: ');
readln(breadth);
erid;
if (reply = 'd') or (reply='D') ther,
begir
write('Eriter the depth of the bearn specified: ');
readlr(depth);
end;
erid;
erid;
cdudl:=breadth*depth*E.5;
cal;{ callirg the procedure to calculate dead load momert {
writeln;
writeln;
if cIflag= true ther,
begir,
x:=0;
y:= lerigth;
writelr('Now please erter the laads');
writelri;
end;
if ssflag= true thers
begin
x:=0;
y:= lergth;
writeln('Now please enter the laads');
writeln;
erid;
if ohflag= true ther,
begir,
flag := true;
while flag = true dg
begin
flag:=false;
write{"Please enter the left support');
write(" of the beam fram left(rats) : ');
meadlr(x);
if {x} lersgtt) or (x<0) then

```
```

                        begin
                            writelri;
                            write{'Left support beyand span');
                    write(' please check ');
                            flag:=true;
                            end;
            end;
        writeln;flag := true;
        while flag = true do
            begin
                flag:=false;
                write('please enter right support (mts):');
                    readln(y);
                if (y)lerigth) or (y(x) ther,
                begin
                if y) lerggth ther writeln(" Right support ');
                write{'exceedsthe beam spar, please check');
                if y<x ther, writelri(" Right support is');
                write("left of left support please check');
                write\' and try again ');
                flag:=true;
                    end;
            erid;
            p:= O;
            for i:= O to 100 do
            tegin
                    c[i,1] := p;
                    p:=p+lemgth;
            ersd;
            writelrs('please eriter the the laads. ');
            erid;
    if frflag= true ther,
begin
x:=0;
y:= lerigth;
writelr;
writelri('Please erter the the laads fram the left.');
write(" Ariy sirikirg of ariy the two supports[y/r,] :");
readln(answere);
if ({answere=' y') or(answere="Y"))then
begin
writelr, "Give the values
af E \& I urict kg/m');
write('E:');readlm(E1);
write("I:');readIn(II);
flag:=true;
while flag=true do
begin
flag:=false:
write("reply[L/R]:');
readln(answerne);
if (ariswere='r') or (ariswere=' R')
or (ariswere= 'L') or
(arswere=" 1") them
write{",')

```
```

                else
            begin
                            write('Sorry try again');
                    writeln;
                    flag := true;
                    end;
            erid;
        write('Enter the sinking in Cms :');
        readlr(srik);
        51 :=6*e1*I1*smk/length/lerigth;
        52 := 51/50;
        z:=51;
        for i:= 1 to 100 da.
        begin
            h[i]:=z;
            z:=z-5ご
        erid;
    if (answere='r') or (ariswere='趷 )then
        for i:=0 ta 100 dc h[i]:=-h[i];
        far i:= O to 100 de writeir(ti[i]);
                    ernd;
        erid;
    p:=0;
    for i:=0 to 100 da
        begir
            c[i,1] := p;
            p:=p+length;
        erid;
    eritry :=true;

```

```

for i == 0 to 100 da
begir,
c[i,6] :=0;
c[i,7] :=0;
c[i,8] :=0;
end;
while eritry =true do
begir
for iz=0 to 100 da begin
c[i,2] :=0;
c[i,3] :=0;
c[i,4] :=0;
f[i] :=O;
g[i] :=0;
erid;
entry := false;
flag:= trueag
while flag = true do
begin
fiag := false;
writelr;
WRITELN("The follawing load opticirs are available');

```
```

        writeln("Type " 1 " for coricentrated load") ;
        writeiri('Type " }2\mathrm{ " for distributed load');
        writelri;
        write('What is your optior, :' );readlri(arswere);
        ;
        if (ariswere ='1') or (ariswere = 'こ')
        then writelr
        else
            begir:
                writeln;
                writelri(" Sorry irvalid aptior, try again');
                flag:= true;
            end;
    erid;
    if ariswere =' 2' then udlflag := true;
if arswere ='1' then
begim
flag:= true;
ptflag:=true;
while flag =true do
begirs
write('Enter the laad from left(mts) :");
readln(distarice);
writelrs;
flag:= false;
if distarice; lergth then
begir
writelri"LGad exceeds the span try agairi");
flag:= true;
erid;
ernd;
write('Magnitude of the laad :');
readln(internsity);
writeln;
g1 := gi + intensity* (y-distarice);
ge := g2 + intensity;
if clflag=true ther,
begin
z:=0; z1:=0;
for* i:=trunc(round (distarnce*100/1ength))
dowrita o da
begin
c[i, Э]:=c[i,9]+z;
z:=z+intersity*length/100;
c[i,1O]:=c[i, 1O]+iritensity;
erid;
for i:=0 to 100 do
begin
c[i,5]:=c[i,9];
c[i,6]:=c[i,10];
Erid;
end;
if (distance } x ) and (distance ( y ) ther,

```
```

    begin
    for i:= trumc(round(distance*100/1ength))+1
                to 100 da
                        c[i,8] := intensity;
    for i:= 0 to 100 do
        begin
            c[i,7] := c[i,7] +c[i, B];
            c[i, 8]:=0;
        end;
    lint := {intensity/100 * (distance-x)*
                    (y-distarce)/(y-x))
                            /((distance-x)/length);
        z:=0;
        for i:=trunc{round(100 *x/length)) to
            trunc(round(distance /length*100)) do
        begin
            c[i,2]:=z;
            z:=z+1 int;
        end;
    Rint := {interssity/100 * {distance -x\rangle *
                                    (y-distance)/(y-x))/
                            ((y-distarce)/lerggh);
    z:=0;
    for I:= trunc(round{100 * y/lerigth)) downta
                            trunc(round(distarice /length*100))do
        begin
            c[i,2] := z ;
            z:=z + rint;
        end;
    end;
    if {distance ( < ) or (distarnce ) y ) them
begin
if distance \& }x\mathrm{ then
begin
for i:=trunc(round (distance*100/length)) +1
to 100 do
c[i,8] := intemsity;
for i:= 0 to 100 do
begin
c[i,7] := c[i,7] +c[i, B];
c[i,8]:=O;
end;
z:=0;
lirit := intersity/100* (x-distarce) /
((x-distarce)/length);
for I:= truric(round (distance /lergth*100))
to trunc(rourid(100* x/1ength)) do
begir
c[i,\Xi]:=-z;
z:=z+lirit;
erid;
writelri(z);
lint := intersity/100*(x-distarice)/

```
```

                                    ((y-x)/lergth);
            z:=0;
            for I:=truric(rourid {100*y/length)) downto
                    trunc(round (100* x/length)) do
                    begin
                c[i,2]:=-z;
                z:=z+lint;
            end;
        end;
        if distarice>y ther
        begin
            for i:=trunc(round (distance*100/1 ergth)) +1
                        to 100 da
                c[i,8] := intensity;
            for i:= 0 to 100 do
                begin
                    c[i,7] := c[i,7] +c[i;8];
                    c[i, 8]:=0;
                    end;
            rint := intensity/100*(distance-y) /
                                    ((distance-y)/length);
        z :=0;
        far I:= truric(rourid{100* distarice/length))
                        dawrito trunc(rourid (y/length*100)) do
                        begir
                        c[i, 己] := -z;
                    z:=z+rint;
                    end;
        z:=0;
        rint := intersity/100*(distance-y)/
                            (<y-x)/lerigth);
        for I:= trunc(round(100*x/lergth)) to
                    trunc(round (y /lerigth*100)) da
            begir
                c[i,2] := -z;
                z:=z+rint;
            end;
        end;
        end;
    if clflag=false then
        begin
            for I:= 0 to 100 do
                begin
                        c[i,4] :=c[i,4] +c[i, 2];
                c[i,5] := c[i;5] +c[i,4];
            end;
        end;
    if fxflag=true then
begin
if ptflag=true then
begin
for i:=trunc(round(distance*100/1ength))+1
to 100 do
c[i,8] := intensity;

```
```

                                    for i:= 0 ta 100 dc
                                    begir
                            c[i,7] :=c[i,7] +c[i,8];
                            c[i, 8]:=0;
                            erid;
                            z1:=iritersity*distarice * (length-distarice) *
                            (length-distarice)/(length*length);
    zE:=iriterisity * distarice * distarice *
(lergth-distarice) / (lerigth*lerigth):
m1:=m1+z1;
m2:=mこ+z2;
z:=m1;
a1:=(m2-m1)/100;
far i==0 to 100 do
begir
g[i]:==z;
z:=z+al;
f1:=g[i]-c[i,5];
f[i] :=-{ f[i]+fi};
erid;
end;
erid;
write("Ariy other laads ?[y/ri] :");
readln(answere); ;writelri;
if ((answere=' (') or(answere=' Y')) then entry := true
else
if cIflag <>true then
begir
entry:=false;
g3 := g1/(y-x);
g4 := घ2 - g3 ;
for i:= truric(rourid (x*100/1ength))+1
to 100 da
c[i,8] := g3;
far i:= trunc(rourid (y*100/1ength))+1
to 100 da
c[i,8] :=c[i, 8] +q4;
sh1:=0;she:=0;
for i:=0 to 100 da
begin
c[i,6] :=c[i,6]+c[i, 8]-c[i,7];
if c[i,6]) she then she:=c[i,6];
if c[i,6]<sh1 then sh1:=c[i,6];
end;
erid;
end;

```
while udlflag =true do
    begin
            flag: \(=\) true;
            while flag =true do
                begir
                    flag := false;
```

    writeln;
    write(' Please give the pairit from left');
    write(" where the udl starts(meters) :');
    readlri(distarice);
    if distance (0 then
    begin
                writelr(* laadirg beyond the limits of');
                write(' beam check ard try again');
                flag := true:
            end;
    end;
    flag:=true;
while fiag =true do
begir
flag := false;
writeln;
writelri;
write{" please give where the udl erids');
write('from left(meters) : ');
readlri(spari);
if spar, >length ther,
begin
write{'The udi spans beyond the beam');
write{" please check ard try agair' ');
flag := true;
end;
end;
writeln;
write{' Please give the intersity of the');
write(' udl(toris/meter) : ");
readlri(irtensity);
if ciflag=true then
begir,
11:=distarice:
1こ:=length/100;
z:=(spari-distarice)*iritensity*
(spar-distarice)/シ;
z1:=iriterisity*lemgth/100;
s1:=(spari-distarme)*iriterısity;
se:=spari;
for i:= truric(round{distarice*100/1ength)) to
trunc (rourid (spari*100/lerigth)) do
begir,
c[i,9]:=c[i,9]+z;
z:=(s己-distance)*intensity
*(s2-distance)/己;
5こ:=5ごー1き;
c[i,1O]:=c[i,1O]+51;
s1 :==51-21;
end;
z:=0; s1:=(spari-distance)*intersity;
for i:=trunc(round(distance*100/length))-1
downto 0 do
begin
z:= 51*(5pan-11)/2;

```
```

                        c[i,9]:=c[i, 3]+z;
                        11:=11-1E;
                        c[i,10]:=c[i,10]+51;
                    end;
                    for i:= 0 to 100 do
                    begir
                        c[i,5]:=c[[i,9];
                        c[i,6]:=c[i,10];
            end;
    end;
    if (span (= y) ther,
    begin
        R1 := { (span-distarice) *iriterisity *
            (y-distance-(spari-distarice)/2))/(y-x);
        zs:={span-distance)*intensity /
            (100/(length/(spar-distarice)));
        zh1:=0;
        g1:=g1+r1;
        for i:=trunc(round(distance*100/1ength)) to
                            truric(rourid(spari*100/lerigth)) do
            begin
                c[i,7]:=zh1;
                zh1:=c[i,7]+zs;
            end;
        for i:=truric (round(spars* 100/1ength)) +1
                to 100 da
            c[i,7]:=2h1;
    erid
    else
begin
if distarnces = y ther
begir
R1:=-({spari-distar,ce)*(spar, -distarice)/2)*
intensity;
z5:=(spari-distarnce)*iritensity /
(100/(length/(spari-distance)));
zh1:=0;
g1:=g1+r1;
for i:=trunc(round (distarice*100/1 ergth))
to trunc(round(spari*100/1ength)) do
begir,
c[i;7]:=2h1;
zh1:=c[i,7]+25;
end ;
for i:=trunc(round(sparn*100/length))
ta 100 da
c[i,7]:=zh1;
end;
if distance<y then
begin
R1:= (( intersity * (y-distance)*
(y-distance) /こ)/
(y-x))-((spari}-y)*(spari- y)**
intensity /2 /(y-x));

```
```

zs:={spar-distarce)*intersity /
(100/(lerigth/(spar.-uistarce)));
zh1:=0;
g1:=g1+r1;
for i:=trunc(rourid (distance*100/length))
to trunc(round (spari*100/length)) da
begin
c[i,7]:=2h1;
zh1:=c[i,7]+25;
end ;
for i:=trunc(rourid (spari*100/1ength))
to 100 da
c[i,7]:=2h1;
end;
erid;
RE := (span - distance )*intersity - R1;
if (ri (0) ar (re< (0) them
begin
if ri<0 then write{' The left support ');
write{"should in dowrward direction');
if R己<0 then write{' The right support ');
write('should dcwriward dimectiom');
write{" Darr't worry it is assumed that the');
write(" support is in opposite directicori');
end; z:=0;
L1 := 100 * distarce/lergth ;
L己 := sparn * 100 flength;
m :=trunc(raurid(11)) ; }n:==\mathrm{ trunc(round(l2));
z:=0;
for i:= m to n do
begin
c[i,E]:= z;
z:= (z+intersity *lerigth/100);
end;
for i }:=n+1 to 100 da
c[i,2] :=interisity * (spari-distarice);
far i := 0 to 100 da
c[i,4] := - {c[i,2]* (C[I,1]/1OO-distar,ce)/2);
for i := trúric(rourid (100**/lerigth))
to trunc(rourid (y*100/1erigth)) do
begin
if c[i,1] ( spar,*100 thers
c[i,4] := ri * (c[i,1]/100-x)
-c[i,2]*(c[i,1]/100-distarce)/ 2
else if c[i,1] )= spar,*100 ther,
c[i,4] := ri * (c[i,1]/100- x)
-c[i,2] *
(c[i,1]/100-distarice-(spar-distance)/2);
erid;
for i := truric(raund(100*y/lerigth)) to 100 da
begir
if c[i,1] ( spari*i00 ther,
c[i,4] := r1 * (c[i,1]/100-x)

```
    : 4
erid;
if ciflag=false ther
for \(i=0\) to 100 da c[i,5]:=c[i,5]+c[i,4];
writeln;
if fxflag=true then
begin
if udlflag=true ther, begir
```

                                    zs:=(spar-distarice)*iritersity /
    ```
(100/(1ength)(spar-distarice))); 2h1: \(=0\);
for \(i==t r u n c(r o u r d(d i s t a r c e * 100 / 1\) ength)) to
truric (raurid (spar*100/lerigth)) do
begirs
c[i, 7]:=2h1;
zh1:=c[i,7]+zs;
erid;
for \(i:=\) truric (rourid (spari*100/length)) to 100 da \(c[i, 7]:=2 h 1 ;\)
```

                z:=0;
    ```
                for \(i:=0\) te 97 do
                        begir
\[
z 1:=z ;
\]
\[
z:=z+(c[i, 5]+c[i+1,5]) / 己
\]
* (length/100);
end;
uarea: =a1;
z:=0;
for \(i:=0\) to 99 da
begin
\(z:=z+(c[i, 4]+c[i+1,4]) / 2 * 1\) erggth/100 *
(c[i, 1]/100+length/200):
z1: = z;
end;
имак: = 1 ; \(z 1:=4 *\) uarea/length-umax*6/length/lerigtio; z2: = 2*uarea/length-z1;
m1:=m1+z1;
m己: =me+z2;
z:=m1;
for \(i==0\) to 100 do begin
g[i]: \(=2\);
\(z:=z+(m 2-n 1) / 100 ;\)
f1:=g[i]-c[i, 5];
\(f[i]:=-(f[i]+f 1) ;\)
end;
end:
\[
\begin{aligned}
& \text { +Re * (C[i,1]/100-Y) } \\
& \text { - c[i, } 2] *(c[i, 1] / 100-d i s t a r i c e) / z \\
& \text { else if } c[i, 1]>=5 \text { par then } \\
& c[i, 4]:=r 1 *(c[i, 1] / 100-x) \\
& +r 2 *(c[i, 1] / 100-y) \\
& \text { - c[i, 2]* } \quad c[i, 1] / 100-d i s t a r c e \\
& \text {-(span-distance)/2 ); }
\end{aligned}
\]
end：
```

    write{" Ariy other loads ?[y/n]: ');
    ualflag:=false;
    readln(answere);;
    if ((answere=' Y')OR(answere=' Y'))then entry := true
        else
            if clflag <> true ther
                begin
                entry:=false;
                    gこ: =qこ+(span-distarice)*iritersity;
                    g3 := ge -g1;
                    far i:= trunc(round (x*100/lerghth)) +1
                    to 100 da
                        c[i,8] := g1;
                    far i:= trunc(round (y*100/1ength))+1
                    to 100 da
                        c[i, B] :=c[i, 8]+g3;
                sh1:=0;she:=0;
                for i:= 0 to 100 de.
                begir
                                    c[i,6]:=c[i,6]+c[i, 9]-c[i, 7];
                                    if c[i,6]) she then sh2:=c[i,6];
                                    if c[i,6](sh1 then shi:=c[i,6];
                                    end;
            erid;
    ```
            end;
        end:
end;
for \(i:=0\) to 100 da \(c[i, 7]:=c[i, 7]+c[i, 5] ;\)
```

11:=0;12:=0;
for i:=0 ta }97\mathrm{ do
begir
if c[i,5]< 11 then 11:=c[i,5];
if c[i,5]> 1こ ther, le:=c[i,5];
end;

```
assign(fil1, 'data.dat');
rewrite(fili);
writelri(fil1, brntype);
writeln(fil1, lergth*1000.0);
writeln(fill; coricgrd);
writeln(fili, grade);
writeln(fil1, 11*10000000.0);
writeln\{fil1, \(12 * 10000000.0\}\);
writeln(fil1, shi*1000000. 0);
writeln(fill, she* 1000000.0 );
writelri(fil1, stldia*10);
writelriffil1, depth*1000.0);
writelri(fill, breadith*1000. 0 );
clase(fili);
```

clrser:
write('Would you like to see the graphical output [y/n] :");
read(ariswere);
if ((answere=' y')OR(answere=' Y'))then
begin
graphcolormode;
r:=0;
for i:= 0 to 100 do
if abs(c[i,5]) ) r ther, r:=abs(c[i,5]);
if r{}}0\mathrm{ then
begir
J:=48;
draw (50,50,248,50,1);
for i:= = to 100 da
begin
1:= trunc(rourid (c[a,5]/r * 40.0));
j:=\jmath+2;
k:=50-1 ;
draw(J,5O, J,k,巳)
erid;
if fxflag=true then
begin
j:=47;
draw (50,30, 250, 30,1);
for i ==0 to 100 do
begin
l:= trunc(round(g[i]/r * 40.0));
\jmath==\jmath+己;
k:=50-1 ;
draw (j,50, J, k;9);
end;
J:=49;
for i:= 0 to 100 do
begin
begin
1:= trunc(round(f[i]/ r * 40.0));
j:= j+己;
k:=90-1 ;
draw(j, Э0, J,k, こ);
end;
end;
end;
r:=0;
for i:= 1 to 100 do
if abs(c[i,6]) } r then r:=abs(c[i,6]);
if r`0 then
begir
j:=48;
draw (50, 160, 248,160,1);
for i:= i to 100 do
begin
l:= trunc{rourd (c[i,6]/r * 40.0));
j:=j+气:
k:=160-1 ;
draw(J, 160, J, K,E)

```
```

                end;
            end;
    write('Like to see the combined graphical [y/n] :");
    read{arswwere);
    if ((answere=' y')OR(answere=' Y'))them
        begin
            clrser;
            graphcolormode;
                r:=0;
                for i:= 0 to 100 do
                    if abs(c[i,9]) ) r then r:=abs(c[i, 7]);
                    if r<<< then
                        begin
                                    J:=48;
                                    draw (50,50,248,50,1);
                                    for i:= 1 to 100 do
                                    begin
                                    1:= trunc{rourid{c[i,9]/r * 40.0)};
                                    j:=3+2;
                                    k:=50-1;
                                    draw(J,50, J,k, こ)
                            end;
                    erid;
        end;
    erid;
    erid;

```
erid.
```

code=2635
nowarrimgs
domaims
file=mf
database
beam{string)
steel(integer)
stldia(real)
miridepth(real)
safe_depth(real)
safe_breadth(real)
Grnfac(real)
length(real)
pbmd(real)
nomd(real)
design_mamerit(real)
pshr(real)
nshr{real)
miribreadth(real)
x_lim(real)
mu_lim(real)
m_laad{real)
pascald(real)
pascalb(real)
clear_depth(real)
finalbreadth(real)
firialdepth(real)
eff_depth(real)
cover(real)
dpth_secticon(real)
concrete(symbal)
comcgrd(real)
stlgrd(real)
predicates
start
wre(real, real,real)
wiridow
option
breadth
dcd
gmi(real,symbol)
gmé(real, symbol)
hm1(real, symbal)
hee(real, sy-bo,)
dcd1(symbol)
dcdE(symbol)
decide
decide_b(real)
decide_d(real)
cvr(real)
bmfactor
rouridit(real, real)
gr{real,real,real)

```
```

less(real, real, real)
coric
b(real)
check(real, real,real)
safe_section(real)
stl
stlrnf(real,real,real)
design
dsgri (real, real, real, meal, real, real, real)
prelim_dsgm(real, real).
max_prsnt_rrf_srigl {real, real, real)
lim_mom_rest(real,real,real)
modfac_tsri(real,real,real, real)
modfac_comp(real,real)
stlcal(real, real, real)
decide_depth(real,real,reai,real,real, real, real)
goal
start.
clauses
modfac_tsri{0.0, 2. 100, 2. 400, 3.000).
modfac_tsri(0.z,1.350,1.600, 2. 300).
modfac_tsri(0.4,1.075, 1.230, 2. 100).
modfac_tsri(0.6,0.750, 1.100,1.730).
modfac_tsri(0.8,0.700,1.025, 1.525).
madfac_tsri(1.0,0. 350,0. 350,1.415).
modfac_tsri{1.2,0.825,0.925,1.320).
modfac_tsri(1.4,0.810,0.900,1. 285).
madfac_tsm{1.6,0.800,0.8BO, 1. 215).
modfac tsrs(1. 8,0.790,0.860,1.150).
modfac_tsri(2.0,0.775,0.840, 1. 125).
madfac_tsriE. 2,0.760,0. 820,1. 100).
modfac_tsr(2.4,0.745,0.800, 1.085).
modfac_tsn{2.6,0.730,0.785,1.065).
modfac_tsri(E.8,0.715,0.770,1.045).
modfac_tsri\3.0,0.700,0.755,1.025).
modfac_comp(0.00,0.00).
modfac_comp (0.25,0.70).
modfac_comp (0.50,1. 14).
madfac_comp (0. 75, 1 _ 20).
modfac_comp(1.00,1. 24).
modfac_comp (1. 25, 1 . 285).
modfac_comp{1.50,1.33).
modfac_comp(1.75,1.36).
modfac_comp {2.00,1.40).
modfac_ccmp(2.e5, 1.42).
modfac_comp (2.50,1.455).
modfac_comp (2.75,1.48).
modfac_comp(3.00, 1. 51).
max_prsrit_rnf_smgl(15, 250, 1. 32).
max_prsmt_rrf_smg!{15,415,0.7E).

```
```

max_prsnt_rnf_sngl(15,500,0.57).
max_prsnt_rnf_sngl(20,250,1.76).
max_prsnt_rmf_sng1(20,415,0.96).
max_prsrit_rrif_smgl(20,500,0.76).
max_prsnt_rnf_5ngl(25,250, 2. 20).
max_prsnt_rnf_sngl(25,415,1.19).
max_prsrit_rnf_smgl(25,500,0.94).
max_prsnt_rinf_sngl(30, 250, 2.64).
max_prsrit_rnf_sngl(30,415,1.43).
max_prsnt_rnf_sngl(30,500,1.13).
lim_mom_rest(15, 250, 2. 24).
1im_mom_rest (15,415,2.07).
1im_mom_rest(15,500,2.00).
1im_mom_rest(20, 250, 2.98).
1im_mom_rest (20,415, 2. 76).
1 im_mom_rest (20,500, 2.66).
lim_mom_rest (25, 250,3.73).
1im_mom_rest (25,415,3.45).
1im_mom_rest (25,500,3.33).
1im_mom_rest (30, 250,4.47).
1im_mom_rest (30,415,4.14).
1 im_mom_rest (30,500,3.99).
start :- write("Lets start"),
opermead(mf,"data.dat"), readdevice(mf),
readlr(Beam),
readreal (LENGTH),
readlri(CC),
readint(SS),
meadreal (NB), Nbmd=1. 5*Nb,
readreal (PB), pbond=Pb*1.5;
readreal(Sh1),S1=Sh1*1.5,
readreal(Sh己),S己=Sh己*1.5,
readreal(Stldia),
readreal (Depth),
readreal(Breadth),
closefile(mf),
asserta(beam (Beam)),
asserta(concrete(CC)),
asserta(steel(SS)),
asserta(nshr(Sh1)),
asserta(pshr(She)),
asserta(lemgth(LENGTH)),
asserta(ribmd (NEMD)),
asserta(pbrnd (pBMD)),
asserta(pascald(Depth)),
asserta(pascalb(Breadth)),
asserta(stldia(Stldia)),window,

```
```

    bmfactor,!, conc,!,stl,!, evr(Stldia),
        safe_section(Lerigth), optiorr, breadth,
        design.
    window :- makewindow(2,1,112, "Auto Beam", 0,0,25,80), n1, n1,n1.
    option :- write{" This expert system by default designs"),
        write{" a singly reinforced beam, if you da not "),
        write("specify dimensiors of the beam section."),
        write("but it opts for a"), rl,
        write{"doubly reinforced beam if the section you"),
        write(" specified cannot safely handle the laads. ").
    conc :- concrete{m10), asserta(concgrd{10)).
    conc :- concrete(m15), asserta(corncgrd (15)).
    conc- : - concrete(m20), asserta(comigrd (20)).
conc :- concrete{m25), asserta(coricgrd (25)).
conc :- concrete(m30), asserta(concgrd (30)).
conc :- cancrete{m35), asserta(corncgrd (35)).
conc :- concrete{m40), asserta(concgrd (40)).
conc :- correrete(m50), asserta(concgrd (45)).
st1 :- steel(1), asserta(stlgrd(500)), A=(805/(1265+500)),
asserta(x lim(A)).
st1 :- steel(2), asserta(stlgrd(415)), A=(805/(1265+415)),
asserta(x_lim(A)).
st1 :- steel(3),asserta(stlgrd(250)), A=(805/(1265+250)),
asserta(x_1im(A)).
bmfactor :- beam(cl), asserta(bmfac( 7.0)).
bmfactor :- beam(5s), asserta(bmfac(20.0)).
bmfactor :- beam(fx), asserta(bmfac(26.0)).
brnfector :- beam(oh), asserta(bmfac(26.0)).
safe_section(L):-bmfac(F), D=L/F,B1=L/E0,BE=sqrt.(L*D/2SO),
gr(B1, B2, B3),write(" safe D,B ", D," ", BJ),ril,
asserta(safe_depth(D)), asserta(safe_breadth(BJ)).
cvr(Dia) :- Dia)ES, asserta(cover(Dia)).
cur(Dia) :- asserta(cover(ごら)).
breadth:-
concgrd(FCK), pascald(D), x_lim(X), stidia(SD), cover(C), phmd(M1),
nond (ME), MJ=abs(M2),gr(M1,MJ,M),
asserta(design_momert (M)), ri.,
write(" There car, be infinite secticms which cari safely "),
write("handle the laads here are few ard opt for ore"), nl, ril,
write(" The moment ir|uced by the specified loading :",M), nl, nl,
write(" Eff_D/Ereadth Clear Depth Ereadth Eff_depth Lim_Moment "),
!, decide_depth(FCK, D,X,SD, C,M,1.5), !.
decide_depth(FCK, D, X,SD, C,M, 1. 5):- !,
nl, BdE=(1.5*M)/{0.36*FCK*(1-(0.4こ*x))*x), D1=1口(Bd己)/3,
DE=exp(D1), D4=D2+C+(Sd/こ), Dm=D4/10,
roundit (Dm,Dn), DS=10*Dr, D6=round (D5-C-(Sd/2)),
D7=round (D5/1.5),
Mom = (0.36*Fck*D7*D6*D6*( 1-(0.42*X))*X),
write(" 1.5 "," ",D5," ",D7," ",DG," MOM), ril,

```
```

decide_depth(FCK, D,X,SD, C,M,S),!.
decide depth(FCK,D,X,SD,C,M, 2):- !,

```

```

D2=exp(D1), D4=DE+C+(Sd/2), Dm=04/100,
Roundit (Dm, Dr), DS=100*Dri,DG=raurid (D5-C-(Sd/2)),
D7=rourd (DG/2), Mom = (0. 36*Fck*D6*D6*DG*(1-(0.42*x))*x/2),
write(" 2.0 "," ",DS," ",D7," ",DE," Mom), ril,
decide_depth (FCK, D, X,SD,C,M, 2. 5),!.
decide_depth (FCK, D, X,SD,C,M, 2.5) :- !,
BdE=2.5*M / (0. 36*FCK*(1-(0.42*X))*X), D1=1n(BdE)/3,
D2=exp(D1), D4=D2+C+(Sd/2), Dm=D4/100,
roundit (Dm, Dri), DS=100*Dr, DG=round (DS-C- (Sd/ट)),
D7=round (D6/2.5), Mom = (0.36*Fck*D6*D6*D6*( 1-(0.42*X))*x/2.5),
write{" 2.5"," ",D5," ",D7," ",DE," Mam), nl,
decide_depth (FCK, D, X,SD, C, M, 3),=!.
decide_depth {FCK, D, X,SD, C,M, 3):- !,
BdE= 3*M / (0.36*FCK*(1-(0.42*X))*X), D1=1n(Bd己)/3,
DE=exp(D1), D4=D2+C+(Sd/2), Dm=D4/100,
rouridit (Dm, Dn), DS=100*Drr, DG=round (DS-C-(Sd/C)),
D7=round (DG/J), Mam = (0.36*Fck*DG*DG*D6*(1-(0.42*X))*X/3),
write(" 3-0"," ",D5," ",D7," ",MG," Marm), ril,
decide_depth(FCK, D, X,SD,C,M,3.5),!.
decide_depth(FCK, D, X,SD,C,M, 3.5):- !,
EdZ= 3.5*M / {0.36*FCK*(1-(0.42*X))*x), D1=1n(Bde)/3,
D己=exp(D1), D4=D2+C+(Sd/2), Dm=D4/100,
roundit (Dm,Dn), DS=100*Dn, DG=rourd(DS-C-{Sd/2)),
D7=round (DG/3.5), MOm = (0.36*FCk*DG*D6*D6*(1-(0.42*X))*X/3.5),

```

```

decide_depth(FcK, D, X, SD, C,M, 4), !.
decide_depth(FCK, D, X,SD, C,M,4):- !,
EdE= 4*M / (0. 36*FCK*(1-(0.42*X))*X), D1=\operatorname{ln}(\textrm{EdE})/3\mathrm{ ,}
Dこ=exp(D1), D4=Dこ+C+(Sd/こ), Drm=D4/100,
roundit (Dm, Dri),D5=100*Dri,DE=rourid{DS-C-{Sd/E)},
D7=round (DG/4), Hem = (0. 36*Fck*D6*D6*D6*(1-(0.4こ*X))*X/4),
write{" 4.0 "," ",DS," ",D7," ",MGM),r,M,
dcd,!.
dcd :-
write(" Do you war,t to specify breadth or depth or"),
write(" both dimensions af the beam secticm [y/ri]:"),
readln(Reply), dcdi(Repily),!.
dcd1(R) :-
R="n",nl,ril,write(" This systems design is based on "),
write{"the IS-4EE codal specificatiors"), safe_depth(D),
safe_breadth(B), prelim_dsgn( }\textrm{B},\textrm{D})
dcdi(R) : - R="y",
write(" Eriter "t" ta specify Breadth"),ril,
write{" Enter "d" ta specify Depth "),nl,
write{" Enter 'ta' to specify Both "),ril,
write(" d/b/ba : "), readln(Reply), dcde(Reply).
dcdE(R) :-
R="d",write(" Eriter the depth[Mm]: "), readreal(D1),ri,
decide_b(D1).
dcde(R) :- R="b",write{" Eriter the breadth[Mm]: "),
readreal(B), decide_d(E).

```
```

dcde(R) :- R="bo",decide.
decide_d(B):-
lerigth(L), desigm_momerit (M), concgrd(FCK), x_lim(X),
stldia(SD), cover(C), safe_depth(Safe),
D = sqrt (M / (0. 36*FCK*B*(1-(0.42*X))*X)),
gr(D,Safe,D1), DE=D1/10, rourdit(DE,D3),
D4=D3*10,D5=D4-C-{Sd/e),
Mom = (0. 36*Fck*DS*D5*B*( 1-(0.4己*X))*X),
write{" The momerit induced by loads : ",M,"N-mm"),nl,
write(" Breadth you specified[Mm] :",B),nl,
write{" Depth which car, sustain the specifed"),
write(" laading[Mm]:",D), nl,
write(" The depth satisfyimg IS-456 ",Safe),nl,
write(" The greater eff-depth ",D1," cosidered ."),nl,
write(" The moment of resisterice :",Mom), rl,Mm=Mom/M,
write(" The factom of safety ",Mm), nl,
write(" Is this section satisfactary to you [y/ri]: "),
readlri(Reply),
Hmi(F,Reply), prelim_dsgri(B,D4).
decide_b(D) :- safe_depth(Safe),S1=Safe/4,D(=S1,
write(" The depth given is not at all safe"),
write("far a sirigly reirifafrced beam.").
decide_b(D) :-1erigth(L), desigr_momerit (M), carscgrd (FCK), x_lim(X),
stldia(SD), cover(C), D1=D-C-(Sd/E), safe_depth(Safe),
B = M / (0. 36*FCK*D1*D1* (1-(0.4こ*X))*X), Bb=B/10,
rourndit (Bb, B1), BE=B1*10,
Bm=L/EO, Bri=sqrt (L*D/2SO),gr(Err, Bri, EJ), gr (BE, B3, B4),
Mom = (0. 36*Fck*D1*D1*E4*{ 1-(0.4こ*X))*X),
write(" The moment induced by lqads : ",M,"N-mm"), nl,
write(" Depth you specified[Mm] :",D),nl,
write{" Breadth which can sustain the specifed"),
write(" laadirg[Mm] :", E), nl,
write(" The breadth satisfying IS-456 ",B4), nl,
write(" The greater breadth is ccosidered ."),ril,
write(" The moment of resistemce : ", Mom), ril,Mm=Mom/M,
write{" The factor of safety ",Mrn),nl,
write(" Is this sectican satisfactory to you[y/n]: "),
readlr(Reply), gml(D1, Reply).
hmi(B,R) : - R="y",
write{" Thank you for concidering my advice."), rl.
hm1(B,R) : - R="n",
write(" Which has to be changed breadth or depth [b/d]: "),
readln(A),hme(B,A).
hmé(E,A) :- A="d",write(" Enter the depth you prefer[mm] :"),
readreal(D), prelim_dsgn(B,D).
hme(E,A):- A="b",dcde("b").
gmi(D,R) :-
R="y",write{" Thanks for corisidering my advice. " ), ril.
gmi(D,R) :- R='"n",
write{" Which Mas to be chariged breadth ar depth [b/d]: "),

```
```

    readln(A), gme(D,A).
    gme(D,A) :-
A="b",write(" Enter the breadth you prefer[mm] :"),
readreal(B), prelim_dsgr(B,D).
gm己(D, R) : - R="d", dcde("d").
wre(B1,B,B2) :- safe_breadth(B),gr(B;B1, B2).
decide :-
pascald(D),write{" The expert is of the opinion that "),
write(" the depth should in no case should be less "),
write("thar, ",D," this depth daesnot include "),
write("modification factor the mimimum depth "),
write{" car be further reduced if the madification "),
write(" factor is considered "), nl, nl,
write(" The expert also warits to advice you that narrow"),
write(" and deep beam are economical and passes"),
write(" greater stiffness and better lateral stability "), nl,
write(" Now please enter your preferable depth and "),
write("breadth use the above data guideliries"), rl,
write(" Eriter the clear depth you prefer = "),
readreal(Reply), b(Reply).
b(D) :- lergth(L), E1=L/60, B2=squrt (L*D/250), gr (B1, B2, B3),
write(" The breadth shculd nct be less than ", E3), nl,
write(" Please eriter the breadth you prefer :"),
readreal (BS), check (B3, BS, D).
check (MD, H, D) :-
B<Mb,write{" This depth vialates IS-455 codal"),
write(" specifications, Try again "),
nl,b(D).
check(M, B, D) : - Bi=D/E, Bi<<.O,
write(" The breadth depth ratio is "),
write("less than 2.0. The secticon may not be ecomomical "),
ril,b(D).
check(Mb,B,D) :- B1=D/E,E1)S,
write(" The depth breadth ratio is toc"),
write{" much please try again"), ril,b(D).
check(Mb,B,D) :-
writef" All the codal specifications regarding "),
write("the depth ard breadth are satisfied"),
l, dsgn(M,Fck, X,Sd, C, B, D).
prelim_dsgr(B,D):-write{B,D).
dsgri(B,D) : -concgrd(F), x_lim(X), design_moment (M),
write(" Effective depth : ",D),ril,
write(" Effective breadth : ",B),nl,
Mlim=0.36*F*B*D*D*{1-(0.4\Xi*x))*x, rl,
write(" Limimting moment of the secticun: ",Mlim), nl,
Mlim3M,
stlcal(D,M,F).
dsgn(M, Fck, X,Sd, C, E, D):-
safe_depth(D), safe_breadth(B), D1=D/10, Bi=E/10,

```

```

    dsgr,(M, Fck, X, Sd, C, E3, D3).
    design:-write(" The desigr, starts.").
    ```
\[
\begin{aligned}
& \text { stlcal ( } D, M, F c k \text { ): }- \text { stlgrd (Fy), } A 1=-F c k * D * D / 2 \text {, } \\
& A \Omega=M * D * F c k / 己 *(0 . B 7 * F y), R 1=(-A 1+s q r t(A 1 * A 1-4 * F y * A 己)) /(2 * F y) \text {, } \\
& R 2=(-A 1-\operatorname{sqrt}(A 1 * A 1-4 * F y * A 2)) /(2 * F y) \text {, } \\
& \text { write(" Rocits of the equation ", R1," ",R2). } \\
& \text { stlrnf( } A, B, D) \text {. } \\
& \text { stlrnf( } A, B, D):- \\
& \text { readln(Reply), Reply="y"; Reply="Y", } \\
& \text { write\{" Enter the dia of bars"), } \\
& \text { readreal (Fai). /* decice the bars and number*/ } \\
& \operatorname{gr}(A, B, C):-A\rangle=B, C=A \text {. } \\
& \operatorname{gr}(A, B, C):-C=B \text {. } \\
& \text { less ( } A, B, C):-A\langle=B, C=A \text {. } \\
& \text { less ( } A, B, C \text { ) : }-C=B \text {. } \\
& \text { roundit }(A, B):-C=\text { rours }(A)-A, C)=0, B=r o u n d(A) \text {. } \\
& \text { roundit }(A, B):-B=\text { round }(A)+0.5 \text {. }
\end{aligned}
\]

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